



# From the measurements with thermocouples to the estimation of heat source terms with inverse methods





# My Team in our laboratory IRDL (21 permanent research professors – 2 engineers – 15 phd students and 2 post doc)

The assemblies

in Lorient : Welding, additive manufacturing, sintering... in Brest: Welding, Bonding...

3 axis: Modelisation, characterisation and instrumentation.

- Multiphysic model (knowledge model for the heat input) and reduced model (for the calculation of mechanical effects: residual stresses and distortions)
- characterisation: definition of the parameters for the simulation
- instrumentation of the experimental characterisation and in situ experiments : thermocouples, infrared camera, speed camera, multispectral pyrometer...





• H. R. B. Orlande, O. Fudym, D. Maillet, R. M. Cotta, Thermal measurements and inverse techniques, CRC Press, Taylor & Francis Group, Boca Raton, 2011.

... and the french engineering techniques.



# Some examples of instrumentation





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## Questions



- Can I define all measurement errors? From:
  - The measurement chain
  - The acquisition system
  - the measurement errors can come from the filtering on the acquisition board. These filters can lead to damping of signal variation.
  - the magnetic fields around the manipulation can lead to signal drift and noise up to +/- 50 ° C. the solution is to remove the wires from the thermocouples perpendicular to the magnetic field and sometimes to shield the wires.



# Questions



- Can I define all measurement errors? From:
  - The Thermocouple calibration
  - Temperature measurement:
    - Why? (History, Temperature: what is it?)
    - How? (principle, characterisation, time constant...)





History ( ref: Metti school – 2009 – Angra dos Reis)



Philon from Byzance (-250 ?) Heron from Alexandria (-200 to + 200 ?) Time of the greek philosophy



The first steam machine The hydraulic pump

Heat and temperature are connected ?



Galien (131 - 210) Greek doctor

Among more than 750 papers on science and medicine, he introduced the concept of "degrees of heat and cold"



Avicenne (980 - 1037) Iranian doctor

Instrument for treating fever, depending on how hot, cold, humidity, dry

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#### Let's build instruments and temperature scales: the beginning of the adventure



#### Problem: fixed points?

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The first theoretical step: 1824 : reflexion on the motive power

is the potential work available from a heat source potentially unbounded?

Can heat engines be in principle improved by replacing the steam by some other working fluid or gaz ?

Towards the second law...

Efficiency is an intuition but

$$\eta = 1 - \frac{f(\theta_1)}{f(\theta_2)}$$



Sadi Carnot (1776-1832)

 $f(\theta) = T$  Lord Kelvin

From Carnot to Kelvin





With Kelvin, the temperature becomes an absolute reality...



William Thomson Lord Kelvin (1824-1907) The kelvin, unit of thermodynamic temperature, is equal to the fraction 1/273.16 of the thermodynamic temperature of the triple point of water



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Temperature





Today,

Temperature scales are based on fixed points

From the first official international temperature scale (ITS) of 1927 to the future...

« Bureau International des Poids et Mesures (BIPM) »

1927 : ITS 27 1948 : ITS 48 1968 : ITS 68 1976 : ITS 76 1990 : ITS 90....



erature (K)	Element Point		
n 3 to 5	helium	vapor	
,803 3	hydrogen	triple	
≈17	hydrogen (or helium)	vapor (or gas thermometer)	
20.3	hydrogen (or helium)	vapor (or gas thermometer)	
.556 1	neon	triple	
.358 4	oxygen	triple	
.805 8	argon	triple	
4.315 6	mercury	triple	
73.16	water	triple	
2.914 6	gallium	melting	
9.748 5	indium	freezing	
05.078	lead	freezing	
02.677	zinc	freezing	
33.473	aluminium	freezing	
234.93	argent	freezing	
337.33	gold	freezing	
357.77	copper	freezing	







Up to 1000°C :

- 1000 ° C - 1200 ° C : Mainly type K thermocouples (40  $\mu$ V / ° C), type N, T ...

- Difficulties with thermocouples: instability, inhomogeneities, temperature limits of 1100-1200 ° C for types K and N

- Up to 1500 ° C, Pt / Pd thermocouples are promising in terms of stability, but they are expensive and their fragility limits them to laboratory use only

- Beyond 1500 ° C, the W / Re family can reach up to 2300 ° C but stability is not clear

New thermocouples are being studied (Ir / Ir-Rh, Pt-20% Rh / Pt-40% Rh)



In use: non-guaranteed material compatibility, especially above 1500 ° C



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Compatibility of materials at high temperature (~ 2000 ° C)

Example: Thermocouple W-Re

Thermocouples W-Re / sheath Mo ⇒+ Pearl ZrO2 - 1750 °C

Thermocouples W-Re / sheath Ta ⇒+ oversheath graphite - 1950 °C

Results after 50h – 1950°C







Association :	Molybdène	Tantale	Graphite pur
SiC	-	-	++
ZrO <sub>2</sub> -8%Y <sub>2</sub> 0 <sub>3</sub>	+	+	+
Graphite pur	-	+	/

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Fixed points at high temperatures : objectives

The realization of the scale above 1000 ° C would be done by interpolation between these fixed points and not by extrapolation as currently: better uncertainties

Applications are as much in optical pyrometry as in contact thermometry. Traceability would be based on fixed points that could even be thermodynamic temperature vectors

Practical means for rapid checking of the stability of measuring instruments at working temperature

In-situ re-calibration of measuring and control instruments

- Completion and study since 2001
- Multiple comparisons
- Thermodynamic phase change temperatures determined
- New implementation of the kelvin HT definition (site www BIPM)

#### International projects: NIST, LNE (France)...











In 1821 T.J. Seebeck observed the existence of an electromotive force, EMF ( $\mu$ V) at the junction formed between two dissimilar metals (Seebeck effect) submitted to a temperature difference ( $T_2 - T_1$ ).

Seebeck effect is actually the combined result of two other phenomenon, Thomson and Peltier effects

Peltier discovered that temperature gradients along conductors in a circuit generate an EMF.

Thomson observed the existence of an EMF due to the contact of two dissimilar metals and the junction temperature.

Thomson effect is normally much smaller in magnitude than the Peltier effect and can be minimized and disregarded with proper thermocouple design.









Seebeck effect

 $E_{AB}(T_2,T_1) = \sigma_{AB}(T_2 - T_1)$ 

 $\sigma_{AB}$  is the Seebeck coefficient ( $\mu$ V.°C<sup>-1</sup>), depends on the two materials A and B





Material	Seebeck coefficient (uV°C <sup>-1</sup> )	Material Seebeck coefficie				
Bismuth	-72	Silver	6.5			
Constantan	-35	Copper	6.5			
Alumel	-17.3	Gold	6.5			
Nickel	-15	Tungsten	7.5			
Potassium	-9	Cadmium	7.5			
Sodium	-2	Iron	18.5			
Platinum	0	Chromel	21.7			
Mercury	0.6	Nichrome	25			
Carbon	3	Antimony	47			
Aluminium	3.5	Germanium	300			
Lead	4	Silicium	440			
Tantalum	4.5	Tellurium	500			
Rhodium	6	Sélenium	900			
K type = 0	Chromel/Alumel	T type = Cop	T type = Copper/Constantan			
σ = 21.7-(-17.3	s) = <b>39</b> μV.°C <sup>.</sup> 1 @ 0 °C	<i>σ</i> <del>=</del> 1.7-(-37.3)	<i>σ =</i> 1.7-(-37.3) = <b>49</b> μV.°C <sup>-1</sup> @ 0 °C			
$\sigma = 400 - (-15) = 415 \mu\text{V.}^{\circ}\text{C}^{-1}$ at 0°C						

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Advantages of thermocouples

- Cheap
- Wide temperature range : 270°C to 2 100°C
- Small: from 0.5µm to.....
- easy to integrate into automated data systems

Disadvantages of thermocouples

- Small signals, limited temperature resolution
- Thermocouples wires have to extend from the measurement point to the readout. Signal generated wherever wires pass through a thermal gradient

- At high temperatures, thermocouples may undergo chemical and physical changes, leading to loss of calibration

- Recalibration of certains types of thermocouples or in certain applications is very difficult







The intrusive effect of the sensor Analyse with an analytic model (J.P Bardon and B. Cassagne) Measurement of surface temperatures

Sources of error in contact measurements: they can be classified into two categories:

- errors related to the measurement of the thermometric phenomenon (seen before);
- errors related to the disturbance of the thermal field caused by the application of the thermometer.

With a thermocouple, we measure the temperature of the thermocouple and not the temperature of the medium



Figure 4 – Transferts de chaleur parasites du milieu vers le thermomètre, et du thermomètre vers l'extérieur

A sensor is attached to a sample. A heat flow is transmitted to the sensor which transmits heat flow through its wires to the external medium. The assembly, sensor and medium, exchanges with the environment by convection and radiation.

All these parasitic transfers induce a local perturbation of the temperature field, positive or negative, depending on whether there is a decrease or increase in transfers from the surface to the outside.

The surface temperature is no longer T but Tp. Moreover, the temperature of the thermometer,  $\theta$ , is generally not equal to this perturbed temperature Tp, because the conditions of contact of the thermometer with the surface, always imperfect, cause its temperature  $\theta$  to differ as much from Tp as the thermal contact resistance rc is high, and the heat flux passing through the sensor-to-surface interface is large.

For measurements in variable regime are added the influences of the differences of thermal capacity between the thermometer and the medium, and of the initial temperatures.



We consider a medium opaque to radiation, large dimension, limited on one side by a flat surface with uniform temperature T. The sensitive element, which we assume to be infinitely thin, is in imperfect contact (contact resistance Rc per unit of apparent surface) with the plane face along a circle of radius y. The connection with the outside is schematized by a bar of known characteristics, of radius  $y_B$  except at the section x = 0, assumed to be circular and of radius y.

The external environment is assimilated to a closed enclosure containing a fluid at temperature Tf which, for simplicity, we assume is transparent to thermal radiation.

It is also assumed that the wall of the enclosure can be assimilated to a black surface (emissivity  $\varepsilon$ =1) at a temperature T<sub>pe</sub> close to T, except for a temperature element T'<sub>pe</sub> >>T

Modeling the steady-state error

Without thermocouple, the surface temperature is T and the temperature of the environment TE

With the sensor, a thermal disturbance causes a transfer of heat flow from the sensor to the outside. Let  $\theta$  be the temperature of the sensor located at x = 0, ie the measured temperature. The measurement error  $\delta \theta = T - \theta$  results from the conjunction of three effects.



Figure 7 - Effet de macroconstriction au sein du milieu

#### Macroconstriction effect in the environment

It is caused by the convergence of the current lines to the measuring zone ( $\pi$  y<sup>2</sup>). It follows that, at the level of this zone, the perturbed temperature Tp is connected to the temperature in the distance T, that is to say at the exact surface temperature before applying the thermometer to it, by the relation T - Tp = r<sub>M</sub>  $\Phi$  (1) r<sub>M</sub> being the macroconstriction resistance; it is due to the convergence of the flux lines towards the contact circle. Its calculation, the hypothesis of a semi-infinite medium is classical; we obtain the two expressions:

$$r_{\rm M} = \frac{1}{4y\lambda}$$
 ou  $r_{\rm M} = \frac{8}{3\pi^2 y\lambda}$ 

depending on whether one assumes the isothermal contact circle or crossed by a flux of uniform density ( $\lambda$  is the thermal conductivity of the medium). The calculation also shows that most of the temperature perturbation is located in the immediate vicinity of the contact circle (94% of the T - Tp drop occurs inside the sphere of center O and radius 10 y).



**Second effect:** Contact resistance effect at the mediumthermometer interface

It is responsible for the temperature drop Tp -  $\theta$  between disturbed temperature and measured temperature. We have Tp -  $\theta$  = rc  $\Phi$ , where rc represents the thermal contact resistance for the area  $\pi$  y<sup>2</sup>: rc = Rc / ( $\pi$  y<sup>2</sup>). This effect is related to the imperfection of the contact, which results from the irregularities of the surfaces. Contact between two solid media occurs only in a number of small areas (1%) between which there remains an interstitial medium.

Figure 8 – Effet de résistance de contact à l'interface milieu-thermomètre

e (m)	25 E-6	2.5 E-6	0.25 E-6	0.025 E-6	0.025 E-6	
k air = 0.025 W.m <sup>-1</sup> .K <sup>-1</sup>						
Rc = e/k	10 <sup>-3</sup>	10-4	10 <sup>-5</sup>	10-6	10 <sup>-7</sup>	
Contact quality	bad				Very good	

#### Third Effect: Effect of fin

- the heat exchanges between the outer part of the thermometer and the ambient medium (between the face x = 0 at  $\theta$ , and the external medium TE) :

 $\theta$  - TE = r<sub>E</sub>  $\Phi$ 

 $r_{\rm E}$  representing the overall thermal resistance between the face x=0 and the external medium. It defines global exchanges with the external environment. It depends in particular on the geometry, the overall surface transfer coefficient h and the thermal conductivity  $\lambda_{\rm E}$  of this external connection.

Conjunction of the three effects

From the three resistances, we deduce the measurement error:  $\delta\theta = K(T - T_E)$  with  $K = \frac{1}{1 + \frac{r_E}{r_c + r_M}}$  (5)

The error committed is therefore proportional to the difference between the temperature to be measured and the equivalent external temperature

the error coefficient K being all the smaller as the sum of the macroconstriction resistances  $r_M$  and the contact resistance  $r_c$  will be small in the resistance of the external connection  $r_E$ .

It should be noted that this model remains valid only as long as the hypotheses that allowed to linearize the radiative exchanges remain verified.

#### Important implications for steady-state contact measurement techniques

- even for perfect contact conditions  $r_c = 0$ , there is an error which depends on the ratio  $r_E / r_M$ . - It will therefore be necessary to ensure that  $r_c$  is as low as possible and stable. The contact pressure must be strong and constant, the surface must be flat, without ripple, the most conductive interstitial medium possible (welding, grease)

- For measurements on an insulation ( $\lambda$  low), r<sub>M</sub> is large and, in general, very much greater than r<sub>c</sub>. The macroconvergence effect will play a major role in the error. It can be reduced by increasing the y-radius of the sensitive element without increasing the subsequent sections of the outer link. A contact disk of good thermal conductivity 1D will be used for this purpose.

#### Example: applications.

Thermocouple measurement with and without contact disc.

Tableau 2 – Valeurs des résistances						
Nature de la paroi	Disque (1)	<i>r</i> M (K.W <sup>−1</sup> )	<b>R</b> c (K.m <sup>2</sup> .W <sup>-1</sup> )	r <sub>c</sub> (K.W <sup>-1</sup> )	<i>г</i> Е (K.W <sup>-1</sup> )	к
Conductrice	sans	5	10 <sup>-4</sup>	127	1 273	0,094
$(\lambda = 100 \text{ W.m}^{-1}.\text{K}^{-1})$	avec	0,25	10 <sup>-4</sup>	0,32	1 293	0,0004
Isolante,	sans	5 000	10 <sup>-3</sup>	1 270	1 273	0,831
$(\lambda = 0, 1 \text{ W.m}^{-1}.\text{K}^{-1})$	avec	250	10 <sup>-3</sup>	3,20	1 293	0,164
(1) $y = 10 \text{ mm}, \lambda_D = \lambda_B.$						

The set of two wires of the thermocouple is assimilated to a bar of uniform circular cross-section with radius  $y_B = 0.5$  mm, infinite length, average conductivity  $\lambda_B = 25$  Wm  $^{-1}$ K<sup>-1</sup> and the heat transfer coefficient h = 10 Wm<sup>-2</sup>.K<sup>-1</sup>. The resistance of this bar is:  $r_B = \frac{1}{\pi y_B \sqrt{2hy_B \lambda_B}}$  (solution of the bar)  $r_E = r_B + \frac{1}{4y_B \lambda_D}$ 

The resistance of the external connection is therefore:  $r_E = r_B$  when there is no contact disk.  $r_E = r_B + \frac{1}{4y_B\lambda_B}$  with contact disc.  $\frac{1}{4y_B\lambda_B}$  representing the resistance due to the convergence of the flux lines from y to  $y_B$  within the disk. It is clear from the table the preponderant influences of  $r_M$  for insulation measurements and  $r_c$  for measurements on the conductor.
### The intrusive effect of the sensor Analyse with an analytic model (J.P Bardon and B. Cassagne)





Figure 13 – Cas où l'application du thermomètre inverse les échanges superficiels

### The intrusive effect of the sensor Analyse with an analytic model (J.P Bardon and B. Cassagne)

Modeling the transient error

For this study, the sensor is abruptly brought into contact with a surface. The sensor, initially at the equivalent uniform temperature TE of the environment, is brought abruptly into contact with the medium assumed to have a uniform temperature T. On the contact circle, the temperature of the medium is Tp (t) instead of T. Due to the imperfection of the contact, the measured temperature  $\theta$  (t), at x = 0 of the bar, is different from Tp.

We suppose that it is still connected to Tp by the classical condition:

$$T_p(t) - \theta(t) = R_c \left( -\lambda_B \frac{\partial \theta}{\partial x} \right)_{x=0}$$
 with  $R_c = r_c \pi y^2$ 

The time evolution of the relative error K (t) defined in the case of three media with different thermal characteristics and for several values of the contact resistance  $R_c$  between the thermometer and the medium.



Figure 21 – Évolution de l'erreur relative *K* (*t*) (brusque mise en contact)

Tableau 6 – Temp	s de réponse <sub>fr</sub> (en	i secondes) (capi	eur mis brusqueme	ent en contact avec	c la surface)	
Nature de la paroi	ν <sub>B</sub>	$R_c/\gamma_B =$				
	(m)	o	0,05 K.m.W <sup>-1</sup>	0,20 K.m.W <sup>-1</sup>	1 K.m.W <sup>-1</sup>	
Conductrice (1)	5 x 10 <sup>-5</sup>	0,012	0,038	0,25	1	
	5 x 10 <sup>-4</sup>	1,8	3,6	20	56	
Isolante à forte capacité thermique (2)	5 x 10 <sup>-5</sup>	1,3	1,32	1,35	1,4	
	5 x 10 <sup>-4</sup>	91	94	101	110	
Isolante à faible capacité	5 x 10 <sup>-5</sup>	2,5	2,54	2,63	2,8	
thermique (3)	5 x 10 <sup>-4</sup>	298	302	307	316	
(1) $\lambda = 100 \text{ W.m}^{-1}\text{.K}^{-1}$ ; $c_{P} = 3,12$ (2) $\lambda = 0,19 \text{ W.m}^{-1}\text{.K}^{-1}$ ; $c_{P} = 1,7$ (3) $\lambda = 0,05 \text{ W.m}^{-1}\text{.K}^{-1}$ ; $c_{P} = 8,4$	× 10 <sup>6</sup> J.m <sup>-3</sup> .K <sup>-1</sup> × 10 <sup>8</sup> J.m <sup>-3</sup> .K <sup>-1</sup> × 10 <sup>3</sup> J.m <sup>-3</sup> .K <sup>-1</sup>					

#### Inertia of measurement

It is characterized by the evaluation of the response time tr defined by:  $\frac{K(tr)-K(\infty)}{K(0)-K(\infty)} = 0.05$ 

K ( $\infty$ ) identifying with the steady-state error coefficient.

this response time depends very much on the characteristics of the medium. For a measurement on a conductor, it depends very much on the contact conditions. It grows very rapidly with Rc, which appears to be the main cause of the inertia of the thermometer.

For a measurement on an insulator, the response times are much greater but practically independent of the contact conditions considered. All this shows that the response time is not a specific characteristic of the sensor but of the medium sensor - environment.





### Break.... Before the applications

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### Analyse the measurement errors



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Objectives:

 Estimation of a source term for a TIG welding experiment – PVR experiment: Programmierter-Verformungs-Riß Versuch (PVR).

Context:

Hot cracking test

Simultaneous realization of a melting line and a tensile stress:

· Constant energy melting line

• Travel speed of traverse at constant acceleration (scanning of a range of high deformation velocity with a single test piece)

Discriminant observations for the segregation craks (liquation + solidification) for quantification of the criteria:

Critical crossing distance and / or speed of occurrence of the first crack







### **ICCS:** Analyse the measurement errors – PVR experiment



- TIG process is stable
  - Easily instrumentable
    - ·Test parameters: U, I, Vs, F
    - $\cdot$  Temperature measurements (Thermocouples)
    - $\cdot$  Arc and Bath Vision
    - $\cdot$  Measurement of distortions
    - Macrography
    - Fusion line without addition of material
    - Low Intensity

Speed camera

Halogen lamps



**COS:** Analyse the measurement errors – PVR experiment



- The thermocouple instrumentation: 14 thermocouples

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**I** Analyse the measurement errors – PVR experiment



- The heat source term : Goldak type - 7 parameters



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$$q(X,Y,Z) = \frac{Q_0.6\sqrt{3}}{abc\sqrt{\pi}^3} \cdot exp\left[-\frac{3(X-X_0)^2}{a^2} - \frac{3(Y-Y_0)^2}{b^2} - \frac{3(Z-Z_0)^2}{c^2}\right]$$

For the estimation, we use 8 thermocouples... With a levenberg Marquardt algorithm.

Conclusion: we find parameters but it is impossible to have a fused zone. Question: Why??????? Philippe Le Masson – IRDL – UBS.





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### **COS**: Analyse the measurement errors – PVR experiment



With the macrograph, we have defined the parameters and we \_ can define a fused zone.



Comparison: theoretical and experimental measurements





## The modelisation of the error (METTI School)

In a steady state case and for a thermocouple on a surface, we have this scheme:

 $\mathcal{E}(t) = T(t) - Tc(t)$ 





### The modelisation of the error

For this case, in the model in steady state, we define three resistances



- With the software « Comsol Multiphysics », we realize a simulation of the welding problem. But, in the second time, we modelise the thermocouples.
- Two configurations are studied:
  - For the first one, the holes for the thermocouples are perpendiculars of the heat flux and the fused zone.
  - For the second, the holes are parallels of the fused zone.
- Moreover, we compute different contact resistances between the thermocouples and the material (Rc= e/ $\lambda$ ,  $\lambda$  = 0.025 W/m/K):
  - Rc=  $10^{-3}$  or  $10^{-4}$  m<sup>2</sup>K/W for a bad contact (e =  $25\mu$ m or 2.5 $\mu$ m)
  - Rc= 10<sup>-5</sup> or 10<sup>-6</sup> m<sup>2</sup>K/W for a mean contact (e = 0.25µm or 0.025µm)
  - Rc =  $10^{-7}$  m<sup>2</sup>K/W for a good contact (e =  $0.0025\mu$ m)

### The modelisation of the thermocouples



### Results of the first estimation

- Analyze the results for a "crime inverse":
  - The criterion decreases
  - After the first iteration, we have:

iteration	Q	Criterion
Initial values	100	187 10 <sup>6</sup>
First	3988	1800
second	4000	0.001

After 2 iterations, we obtain the good value Q = 4000

with "Comsol Multiphysics" the two configurations. Parallele configuration Perpendicular configuration



We execute these configurations with different contact resistances and we use the thermogrammes in the first optimisation loop with a direct problem without the thermocouples.

With this work, we can underline:

- The measurement errors
- The estimation errors of Q

Visualisation of the measurements errors for perpendicular thermocouples RC=0 TC1 RC=1e-3 TC1 RC=1e-4 TC1 RC=1e-5 TC1 RC=1e-6 TC1 Théo TC1 

Visualisation of the measurements errors and comparisons between the two configurations





Conclusions for the two configurations

- 1- With the thermocouples in an isotherm, we have less errors.
- 2- It's very important to have a good contact between the thermocouple and the material
- 3- We must define correctly the space domain to have the less errors.

for 7 iterations	RC	= 1e-3	RC=	= 1e-4	RC=	= 1e-5	RC=	= 1e-6	RC	C= 0
TC perpendicular	2907	37,60%	3788	5,60%	3923	1,96%	4064	1,57%	4061	1,50%
TC parallel	2600	53,85%	3854	3,79%	3987	0,33%	4010	0,25%	3985	0,38%

Conclusions for the two configurations

- 1- An estimation which don't take into account the real instrumentation leads to an error.
- 2- This error can be higher if we have bigger thermocouples (here the diameter of the wire is 50µm). It's impossible to define the characteristic time for the thermocouple. In fact, we study the interaction between the thermocouple with the domain
- 3- At last, if we use thermocouples, we must analyze the transfer between the thermocouple and the material (Rc and heat transfer coefficient between the wires and the environment). And, we have to try to use a real experimental direct problem in the optimization loop. Or eventually, we must quantify the measurement corrections

### NET TG4 : general hypothesis

#### Specimen geometry considered in the simulation

Plate 194 × 150 × 18 Slot 80 mm long ; 6 mm deep

#### Material :

316L austenic steel

#### Welding parameters :

GTAW – 3 weld passes	Pass 2	Pass 3	
U = 10 V	U=10 V	U= 10V	
l = 150 – 180 A	I= 204 A	I = 196 A	
V = 1.667 mm/s	V=76.2mm/mn (1.27mm/s)	V=76.2mm/mn (1.27mm/s)	
E = 0.7 to 1.0 KJ/mm	1.675 KJ/mm	1.768 KJ/mm	
Interpass T° < 80 ℃	T < 60℃	T < 60 ℃	







Multiphysics:





Goldak model



**CIN model** 



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### Heat input : hypothesis

**\rightarrow** Double ellipsoid Q = f (X,Y,Z,t) over the weld deposit and the test piece



$$q(x, y, z) = Q_0 \cdot \frac{6\sqrt{3}.f_{\xi}}{a_{\xi}.b.c.\pi^{3/2}} \exp(\frac{-3x^2}{a_{\xi}^2}) \cdot \exp(\frac{-3y^2}{b^2}) \cdot \exp(\frac{-3z^2}{c^2}) \qquad \qquad \int_V q(x, y, z) \cdot dV = P_0$$

With this function: 7 parameters:  $a_t$ ,  $a_r$ , b, c,  $Q_0$ ,  $f_f$  and  $f_r$ . Moreover,  $f_f + f_r = 2$  and  $a_t(2 - f_f) = a_t f_f$ 

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4th november 2008

-For the numerical resolution, we use:

- -The software "Comsol Multiphysics" for the direct problem
- -Matlab for the inverse problem (Levenberg Marquardt Algorithm)



#### Solution for the pass1 in a quasi steady state simulation

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### Gauss-Newton or Levenberg-Marquardt Method

$$\mathbf{P}^{k+1} = \mathbf{P}^{k} + \left[ \mathbf{J}^{\mathrm{T}} \mathbf{W} \mathbf{J} + \lambda^{k} \mathbf{\Omega}^{k} \right]^{-1} \left\{ \mathbf{J}^{\mathrm{T}} \mathbf{W} \left[ \mathbf{Y} - \mathbf{T} \left( \mathbf{P}^{k} \right) \right] \right\}$$

where  $\lambda^k$  is a parameter and  $\Omega^k$  a diagonale matrix (= I for example)

For uncorrelated measurements

The sensitivity matrix

 $\mathbf{J} = \begin{bmatrix} \frac{\partial \mathbf{T}^{\mathrm{T}}}{\partial \mathbf{P}} \end{bmatrix}^{\mathrm{T}} = \begin{bmatrix} \frac{\partial \mathrm{T}_{1}}{\partial \mathrm{P}_{1}} & \frac{\partial \mathrm{T}_{1}}{\partial \mathrm{P}_{2}} & \frac{\partial \mathrm{T}_{1}}{\partial \mathrm{P}_{3}} & \cdots & \frac{\partial \mathrm{T}_{1}}{\partial \mathrm{P}_{\mathrm{N}}} \\ \frac{\partial \mathrm{T}_{2}}{\partial \mathrm{P}_{2}} & \frac{\partial \mathrm{T}_{2}}{\partial \mathrm{P}_{2}} & \frac{\partial \mathrm{T}_{2}}{\partial \mathrm{P}_{3}} & \cdots & \frac{\partial \mathrm{T}_{2}}{\partial \mathrm{P}_{\mathrm{N}}} \\ \vdots & \vdots & \vdots & \vdots & \vdots & \vdots \\ \frac{\partial \mathrm{T}_{\mathrm{I}}}{\partial \mathrm{P}_{\mathrm{I}}} & \frac{\partial \mathrm{T}_{\mathrm{I}}}{\partial \mathrm{P}_{2}} & \frac{\partial \mathrm{T}_{\mathrm{I}}}{\partial \mathrm{P}_{3}} & \cdots & \frac{\partial \mathrm{T}_{1}}{\partial \mathrm{P}_{\mathrm{N}}} \end{bmatrix}$  $1/\sigma_1^2$  $1/\sigma_2^2$ W = $J_{ij} \cong \frac{T_i(P_1, P_2, \dots, P_j + \varepsilon P_j, \dots, P_N) - T_i(P_1, P_2, \dots, P_j - \varepsilon P_j, \dots, P_N)}{2\varepsilon P_i}$ 

For the definition of the measurement points:

- We can analyse the evolution of the reduced sensitivity coefficients to define the best positions for the measurement points and analyze if the parameters are correlated over the whole time range of study
- analyze the determinant of \_ the matrix [J<sup>T</sup> J]to define the final measurement time

#### Instrumentation



For each estimation, we put thermocouples on two isotherms. In this case, we can obtain the heat distribution in the depth, in the transversal direction and in the longitudinal direction (time)

With the simulation and a comparison with micrographs, we define the instrumentation



#### Instrumentation

For the measurements with thermocouples, we make holes in the specimen. We have a perturbation in the heat diffusion. So we must put the thermocouple wires in an isotherm for a length that around ten diameter. For this experience, we had cut the specimen in three parts





#### Instrumentation

We had 23 thermocouples. These thermocouples were welded in holes: depth 7mm, diameter: 0,65mm. The wires were protected by alumina tubes.



### NET TG4 : general hypothesis

#### Welding parameters :

		PASS 2	PASS 3
	JOB NO	0	0
WELD NO		0	0
	RUN NO	0	0
	WELDER	"G LITTLE"	"G LITTLE"
	DATE	22-Oct-09	22-Oct-09
	TIME	10:27 AM	11:24 AM
	CURRENT	185	180
	VOLTS	9,8	9,9
١	WIRE SPEED	0,1	0,0
TRA	AVEL SPEED	0,0	0,0
	ARC TIME	73,9	81,6
WELD LI	ENGTH (mm)	80	82
ENERGY (kJ)		134	145
HEAT INPUT (J/mm)		1675	1768
GAS FLOW (Lt/Min)		0,0	0,0
INTERPASS TEMP		0,0	0,0
WIRE CONSUMED (mt)		0,0	0,0
	GAS TYPE	0,0	0,0

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## Exploitation







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## Welding axis



## Welding axis



### Pass1






### Pass2





Pass3

## Hypothesis for the simulation

- equivalent heat sources
- The thermo physical characteristics are taken in function of the temperature (ρ; cp; k).
- In the fused zone, we impose an equivalent conductivity to take into account the displacement of the metal: k = 200 W/m.K
- The fused zone is defined around 1450°C for the comparison with the macrograph
- On the surface, we define a heat transfer coefficient:

 $h_{eq} = hcv + \epsilon.\sigma.(T^2 + Tamb^2) * (T + Tamb)/here \epsilon = 0.8$  (the emissivity), and the parameter hcv (heat convective coefficient) is estimated.

## We use only the second interface for



### 1 pass

• Goldak heat source:  $Q_{av}(x, y, z) = \frac{\eta 6\sqrt{3}}{ab_{fo}c\pi^{3/2}} f_{fo}Q.e^{-3\left(\frac{y}{a}\right)^2} e^{-3\left(\frac{x}{b_{fo}}\right)^2} e^{-3\left(\frac{z}{c}\right)^2}$   $Q_{r}(x, y, z) = \frac{\eta 6\sqrt{3}}{ab_{r}c\pi^{3/2}} f_{r}Q.e^{-3\left(\frac{y}{a}\right)^2} e^{-3\left(\frac{x}{b_{r}}\right)^2} e^{-3\left(\frac{z}{c}\right)^2}$ 

The standard deviation = 13°C

	1 Pass
Parameter	Values
a (mm)	5.894
c (mm)	1.5
b <sub>r</sub> (mm)	11.8977
b <sub>f</sub> (mm)	3.719
$f_{fo}$	1.143726
f <sub>r</sub>	2-f <sub>fo</sub>
η	0.8849
hcv (W/m².k)	115.1775







## 2nd pass

Truncated Golda	k heat source:
for $z > H$ $Q_{av}(x, y, z) = \frac{\eta 6\sqrt{3}}{ab_{fo}c\pi^{3/2}}$	$f_{fo}Q.e^{-3\left(\frac{y}{a}\right)^2}e^{-3\left(\frac{x}{b_{fo}}\right)^2}e^{-3\left(\frac{z}{c}\right)^2}$
$Q_r(x, y, z) = \frac{\eta 6\sqrt{3}}{ab_r c \pi^{3/2}}$	$\frac{1}{2} f_r Q. e^{-3\left(\frac{y}{a}\right)^2} e^{-3\left(\frac{x}{b_r}\right)^2} e^{-3\left(\frac{z}{c}\right)^2}$

The standard deviation = 15°C

	2nd Pass
Parameter	Values
a (mm)	
	8.92
c (mm)	4.992
b <sub>r</sub> (mm)	9.71
b <sub>f</sub> (mm)	5.12
$f_{fo}$	1.0866
f <sub>r</sub>	2- <i>f<sub>fo</sub></i>
η	1.42062
hcv (W/m².k)	103.58
H(mm)	14.681





## 3th pass

Annular heat source with gaussian source in the center:

$$Q = \left[\frac{eta_{\_A} * P}{\pi * R^2 * e} * e^{\left[\frac{rt^2}{ra^2}\right]} + \frac{eta_{\_B} * P}{\pi * R^2 * e} * e^{\left[\frac{(rt-b)^2}{rb^2}\right]}\right] * e^{\left[\frac{x^2}{rc^2}\right]} * (z > H)$$



	3th pass
Parameter	Values
e (mm)	0.5
ra (mm)	1.2
rb (mm)	1.16
rc (mm)	4
b (mm)	3.95
eta_A	0
eta_B	0.75
R (mm)	2.149

The standard deviation = 9.5°C





## Conclusions

With the real positions of the thermocouples, the measured temperatures and the macrographs (fused zones), we can obtain the equivalent heat sources.

For the first and second pass, we obtain the shapes of the fuzed zones with the Goldak model (or a truncated model) and we have a good agreement between the measured and simulated temperatures. But for the third pass, where the fuzed zone is extended, it is more difficult to use this model. We use an annular model.

Now, with these equivalent heat sources, we can simulate the mechanical effects... One point stays on the definition of the heat source at the beginning and at the end of the weld.....

#### flyer.pdf

The electron beam welding : the principle



#### Numerical simulation of EB welding of 18MND5 steel



#### The electron beam welding : the simulation

In our problem :

1. The keyhole is defined as a volumic source

- 2. A conductive model for the fused and vapor zones
- 3. metallurgical transformations in the heat affected zone



The direct problems

$$\rho(T) Cp(T) \frac{\partial T}{\partial t} = \frac{\partial}{\partial x} \left( \lambda(T) \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left( \lambda(T) \frac{\partial T}{\partial y} \right) + \frac{\partial}{\partial z} \left( \lambda(T) \frac{\partial T}{\partial z} \right)$$

$$+ \frac{\partial P_{\alpha}}{\partial t} (x, y, z, t) L(T) \left( \rho \frac{\partial H}{\partial t} \right) S(x, y, z, t)$$

$$P(T) = \sum_{phases} P_{P} \rho_{I}(T)$$

$$L(T) = \left( \rho_{\gamma} H_{\gamma} - \rho_{\alpha} H_{\alpha} \right)$$

$$DH = 2.59 \ 10^{5} \ J/Kg \ 1450 \ ^{\circ}C < T < 1550 \ ^{\circ}C}$$

$$DH = 6.59 \ 10^{6} \ J/Kg \ 2600 \ ^{\circ}C < T < 2700 \ ^{\circ}C}$$
Koistinen Marburger

Koistinen-Marburger

$$P = P_{max} \left\{ 1 - exp \left[ -b(T - M_s) \right] \right\} \quad \text{pour } T < Ms$$



$$\begin{split} \eta &= 0.9, \ U = 60 \ \text{kV}, \ I_b = 0.29 \ \text{A}, \\ V &= 2.5 \ \text{mm/s}, \ h = 0.071 \ \text{m}, \\ W_0 &= 0.15 \ \text{mm} \ (\varPhi_0 = 0.6 \ \text{mm}), \\ z_e &= 0.041 \ \text{m}. \end{split}$$

$$S(x, z, t) = \frac{\eta U I_b}{2\pi\omega_0^2 h} \exp(-\frac{x^2 + (Vt - y_s)^2}{2\omega_0^2})$$

$$\xi = Vt$$

$$S(x,\xi) = \frac{\eta U I_b}{2\pi\omega_0^2} \exp(-\frac{x^2 + (\xi - y_s)^2}{2\omega_0^2})$$

The direct problem: the transversal plane



$$\rho(T) Cp(T) \frac{\partial T}{\partial t} = \frac{\partial}{\partial x} \left( \lambda(T) \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial z} \left( \lambda(T) \frac{\partial T}{\partial z} \right) + \frac{\partial P_{\alpha}}{\partial t} (x, z, t) L(T) - \rho \frac{\partial H}{\partial t} + S(x, z, t)$$
$$T(x, z) = T_0 = T_{inf} \quad P_{\alpha}(x, z, 0) = 1$$

#### The data

The phase change enthalpy  $L_{\alpha\gamma}(T) = \rho_{\gamma}H_{\gamma} - \rho_{\alpha}H_{\alpha}$  for the metallurgical transformations

The phase change enthalpies « solid-liquid and liquid-vapor »

- **The emissivity**  $\epsilon = 0.8$
- $\Box$  The initial and external temperatures T<sub>0</sub>=20°C
- The initial and finishing transformation temperatures of all phases (C.C.T. diagram of 18MND5 steel)
- The thermophysical characteristics  $\lambda(T)$ ,  $\rho(T)$  et C(T) of the metallurgical phases.
- The source term:

$$S(x,z,t) = \frac{\eta UIb}{2\pi\omega_0^2} \exp(-\frac{x^2 + (Vt - y_s)^2}{2\omega_0^2}) \frac{2}{h} \left(1 - \frac{z}{h}\right)$$



The direct problem: the longitudinal plane

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The 2D quasi steady longitudinal problem



The direct problem: the longitudinal plane



- ✓ The developed codes are validated with regard to commercial codes.
- ✓ Differences exist however between the experimental and calculated kinetics
- $\checkmark$  The estimation of a source term is thus interesting.

The inverse techniques

✓ Parameter estimation

✓ Function estimation

 $J(S) = \frac{1}{2} \sum_{i} \left[ T(\overline{x}_{i}; S) - f_{i} \right]^{2}$ 

- 1. The Levenberg Marquardt method
- 2. The iterative regularisation method





The Levenberg Marquardt method

The linear model in the 2D quasi steady longitudinal plane



$$S(x, y) = \frac{P_W}{W_{FE}^2} \exp\left[-\frac{x^2 + (y - y_s)^2}{W_{FE}^2}\right]$$
$$\overline{P} = \{P_W, W_{FE}, y_s\}$$
$$P_W = \frac{2}{h} \left(1 - \frac{z_e}{h}\right) \frac{\eta U I_b}{\pi} \quad W_{FE} = \sqrt{\frac{\Phi_0^2}{8}}$$
avec  $P_W = 60000 \text{ W/m}, y_s = 15$ mm, et  $W_{FE} = 0,353 \text{ mm} (F_0 = 1 \text{ mm})$ 

The Levenberg Marquardt method:

Sensitivity analysis



The Levenberg Marquardt method :

#### Sensitivity analysis





# estimation with the iterative regularization method and with the conjugate gradient method



# estimation with the iterative regularization method and with the conjugate gradient method




# estimation with the iterative regularization method and with the conjugate gradient method



#### Conclusions for the theoretical estimation

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The distribution of the source according to the direction of advance of the beam and the direction of depth are well estimated in spite of a very small Fourier number.

The distribution according to the transverse direction cannot be estimated, but the sum of energy is well found.

## The instrumentation

# preliminary tests





#### Informations with the preliminary tests

a) We use thermocouples type K : diameter 50  $\mu$ m with T < 1267 °C.

b) The thermocouple installation is made starting from the shape of the reference welding on which we can raise the molten and heat affected zones, corresponding respectively to the isotherms 1500 °C and 723 °C.

c) We sought to position the thermocouples around this isotherm  $1200^{\circ}C$  (+/- 0.3 mm).

d) We chose to carry out two samples of 3 blocks having 2 interfaces equipped with thermocouples. Let us note that the interfaces red (23 thermocouples) and blue (24 thermocouples) are on the sample 1 and the interfaces black (23 thermocouples) and green (23 thermocouples) on the sample 2. We have 93 thermocouples in the heat affected zone.

# The experiment







#### Welding of the samples

Welding was carried out on the site of the establishment of Indret of the Management of the Shipbuildings (D.C.N. Propulsion)

The maximum power of the electron beam is 100 KW for a vacuum chamber of 800 m<sup>3</sup>

 $U = 60 \text{ kV}, I_{h} = 0,29 \text{ A}, If = 2.46 \text{ A}, V = 2,5 \text{ mm/s}$ 

After welding, 80% of the thermocouples gave exploitable information.

A checking of the technological choices was carried out by recutting the samples and by precisely determining the position of the thermocouples compared to the beam axis



#### temporal synchronization of the 4 interfaces



## Results for 52 thermocouples taken out of the 4 interfaces Levenberg Marquardt Method for $W_{FE}=0.000353$ ( $\Phi=1$ mm).



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## Results for 52 thermocouples taken out of the 4 interfaces The iterative regularisation method



#### results for 52 thermocouples taken out of the 4 interfaces





# results for 52 thermocouples taken out of the 4 interfaces comparison



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Definition of a optimal source

An optimal source was defined in order to set up a Gaussian distribution in the side direction where the estimate is not correct

$$S'(x) = \frac{2 \iint S(x,z) dx dz}{\Delta H \sqrt{2\pi}\omega_0} \exp(-\frac{x^2}{2\omega_0^2})$$

#### Comparison between the optimal and estimated sources



#### Comparison between the optimal and estimated sources





# VI Conclusions

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In this work, we have compared two methods for the estimation of the source term. (Iterative Regularization method and Levenberg Marquardt method)

A theoretical study has been realised

An experiment has been defined

The experimental estimation gives an optimal source. The heat affected zone limit is correct but the fused zone limit is not correct.

# Thanks for your attention