

Experimental identification of transfer functions
for diffusive and/or advective heat transfer
for linear time invariant dynamical systems

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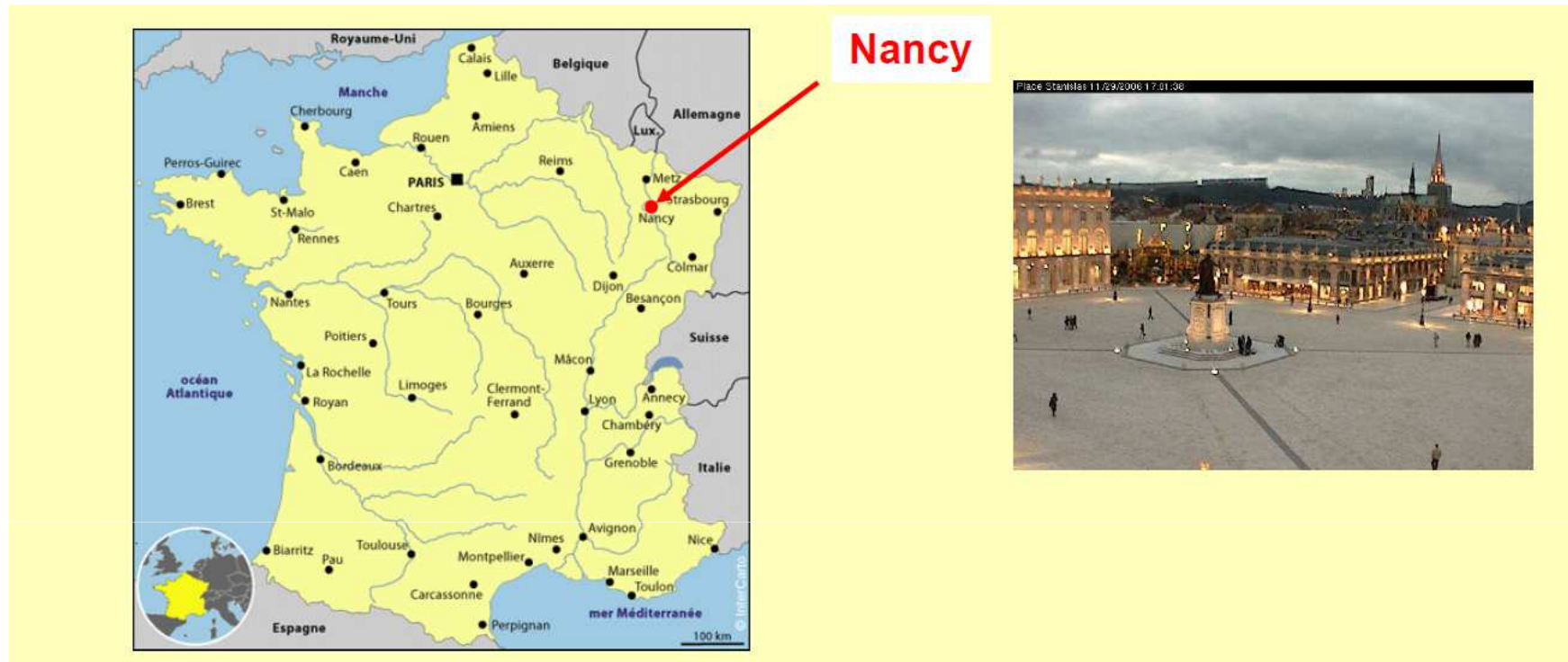
Laboratoire d'Energétique et de Mécanique Théorique et Appliquée (LEMETA)

New Trends in Parameter Identification for Mathematical Models

IMPA, Rio de Janeiro, Brazil, October 30 – November 3, 2017

Contribution: Waseem Al Hadad





Experimental inverse problems in heat transfer and engineering
 METTI Group, SFT (French Heat Transfer Society)

Recently: interest in **convolutive models** and associated **inverse problems**

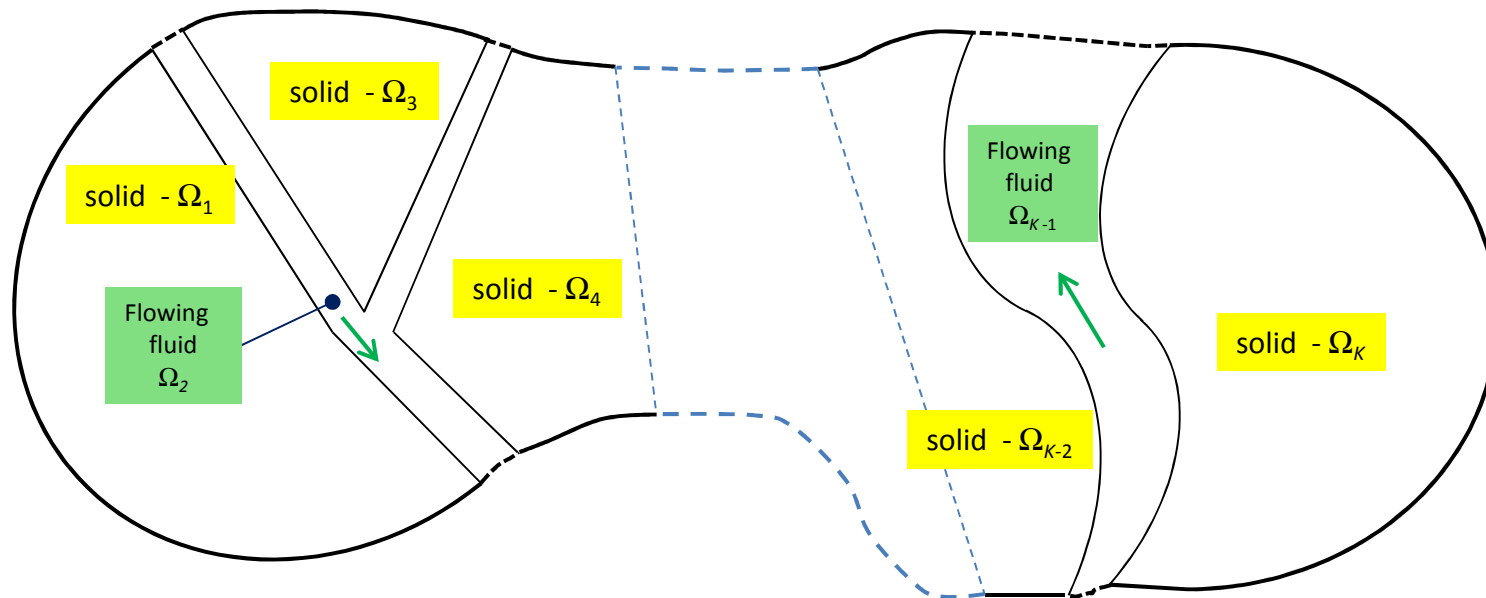
- * Pollutant source identification in a ventilated domain (turbulence, transient concentration measurements)
- * Transient thermal behaviour of heat exchanger (PhD W. Hadad, Fives Cryo postDoc)
- * Virtual sensor construction in a furnace under vacuum conditions (PhD T. Loussouar, Safran Group)

Scope

1. **Forced thermal response of Linear advective/diffusive systems with Time Independent (LTI) coefficients**
2. **The calibration problem**
 - 2.1 **Case of a heat exchanger**
 - 2.2 **Experimental Impedance/transmittance estimation for a half heat exchanger**
3. **Analysis of deconvolution deadlocks**
 - 3.1 **Reference case: 1D transient conduction**
 - 3.2 **Noisy matrix and Total Least Squares**
 - 3.3 **Comparison of calibration methods**
4. **Rectangular deconvolution**
 - 4.1. **Point versus averaged values for input and unknown**
 - 4.2 **Rectangular deconvolution (using « stairs » parameterizing)**
 - 4.3 **Rectangular estimation with $n < m$ non uniform (NU) time steps**
5. **Conclusions/perspectives**

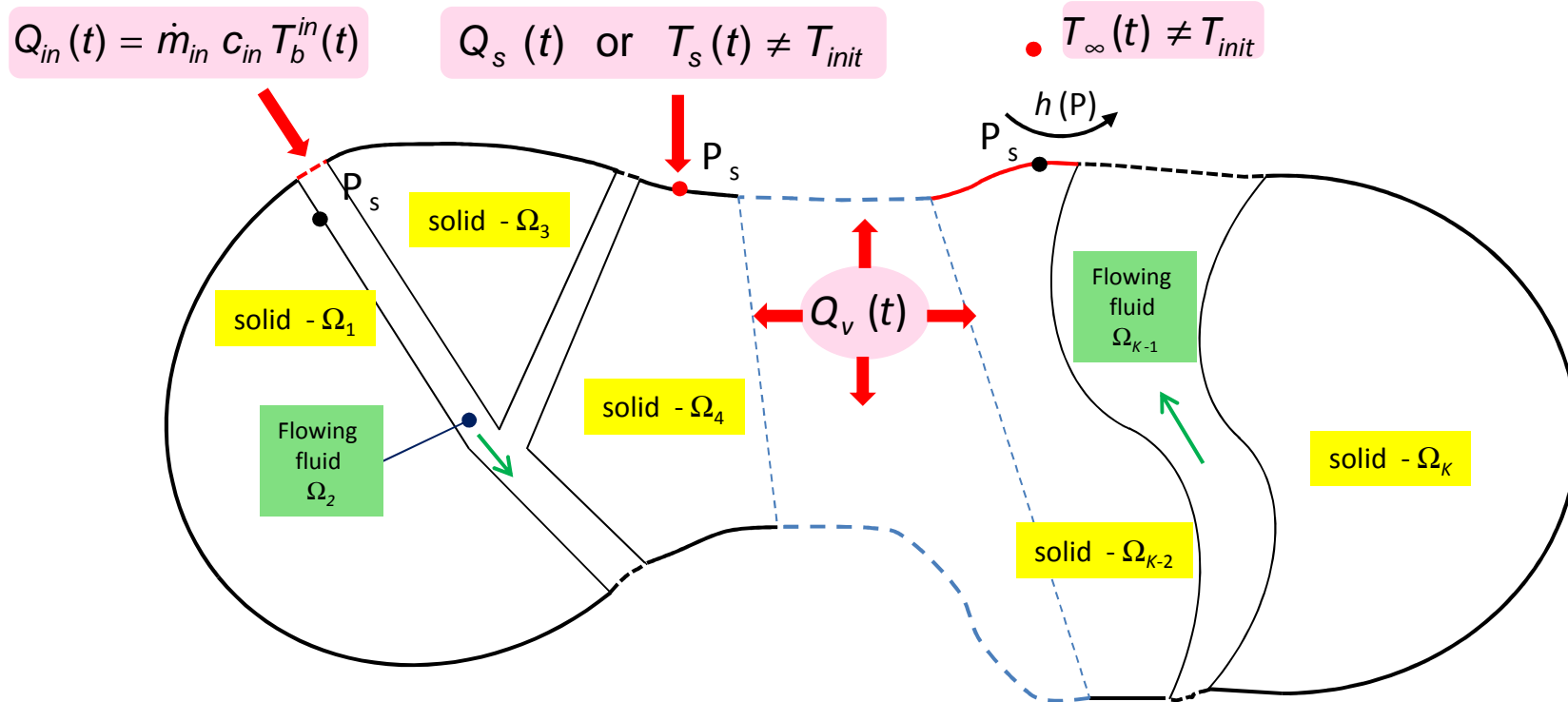
1. Forced thermal response of an advective/diffusive system with time constant coefficients

Material **multicomponent** system = K **solid** or **fluid** domains



Assumptions: **time constant thermophysical properties** and **velocity field**

Initial **uniform state** or **steady state** temperature field
 + one **single separable** thermal excitation



Time part of thermal excitation $u(t)$ (starts at time $t = 0$):

- volumetric heat source $Q_v(t)$
- surface heat or temperature source $Q_s(t)$ or $T_s(t)$
- change of external fluid temperature $T_\infty(t) \neq T_{init}$
- change of temperature at one fluid inlet $T_b^{in}(t)$

Fixed geometrical support:

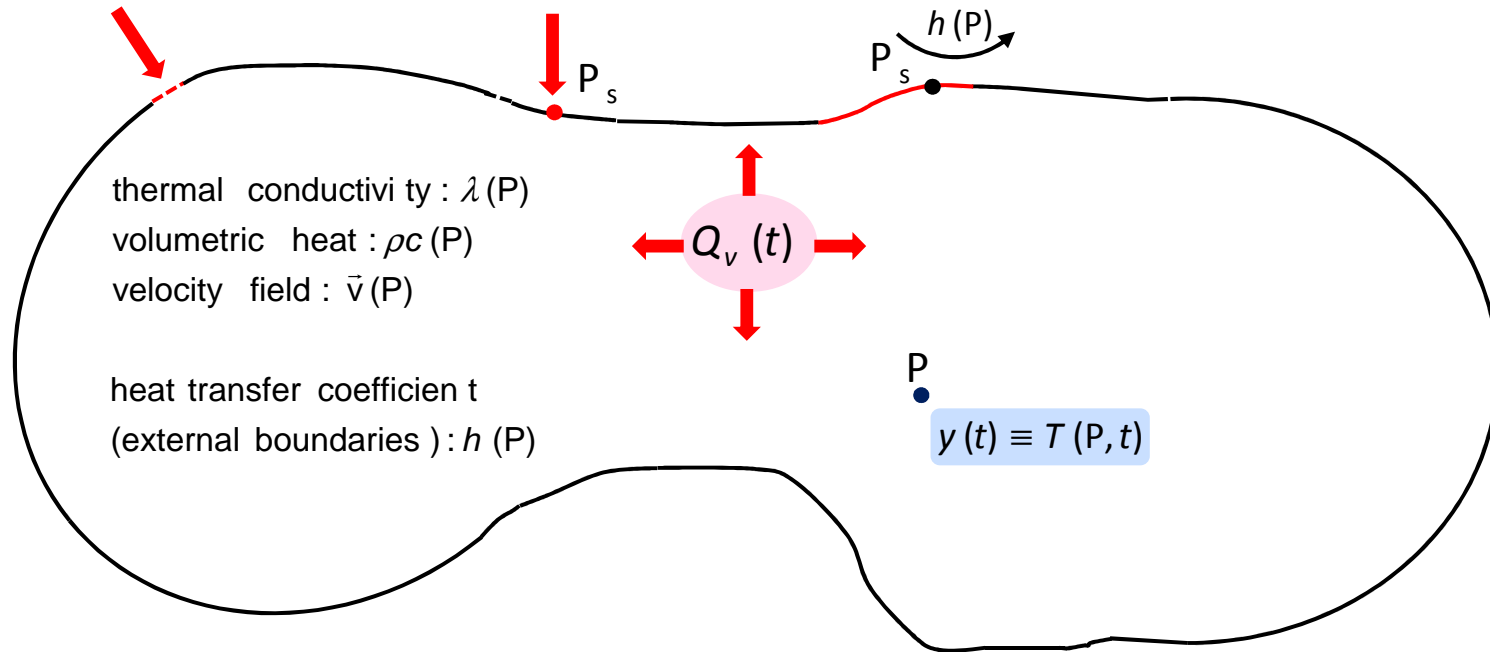
- point
- line
- surface
- volume

Change of perspective: one single **heterogeneous fluid** in **one single domain**
 (if solid part : zero velocity)

$$Q_{in}(t) = \dot{m}_{in} \dot{c}_{in} T_b^{in}(t)$$

$$Q_s(t) \text{ or } T_s(t) \neq T_{init}$$

$$T_{\infty}(t) \neq T_{init}$$



Transient separable thermal excitation :

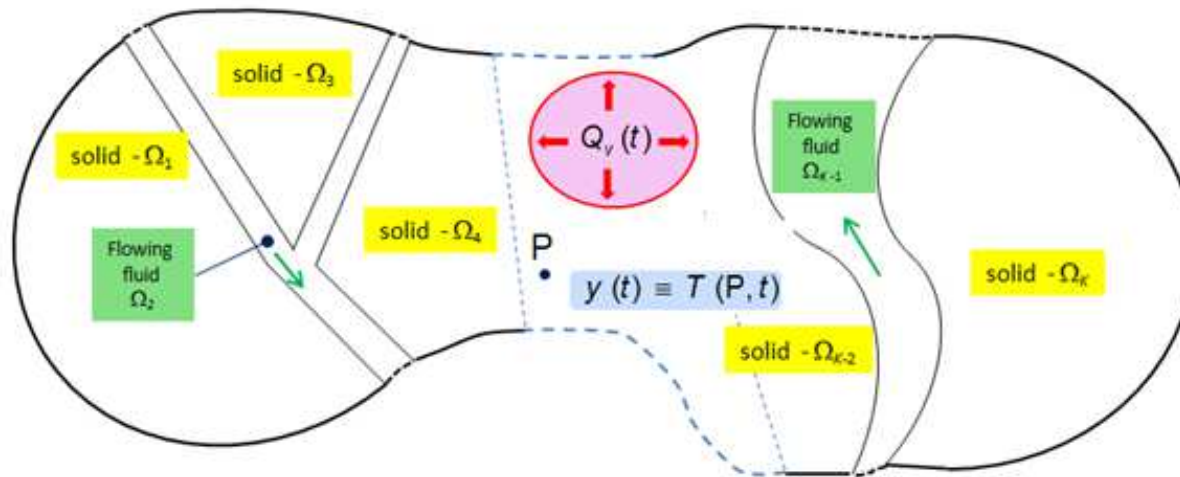
Point response at any point P :

$$u(t)$$



$$y(t) \equiv T(P, t)$$

Recap:



Physical system:

Set of solids **AND** fluid(s):

3D forced convection with constant velocities (**in time** but **not in space**)

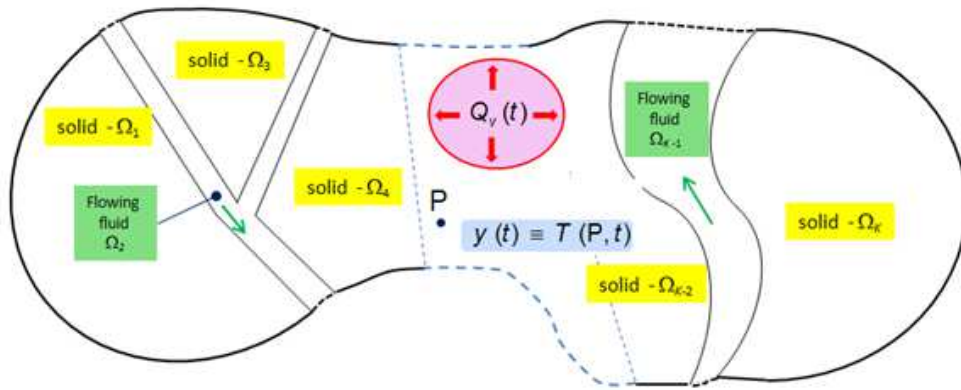
$P = \text{ANY}$ point in the system

One single thermal excitation defined by its support

Assumptions : Transient heat equation + boundary conditions with time-invariant coefficients + uniform initial temperature **or** steady state (the system is **Linear** and also **Time-Invariant** LTI)

$$\boxed{\rho c(P) \frac{\partial T}{\partial t}(P, t)} + \boxed{\rho c(P) \vec{u}(P) \cdot \vec{\nabla} T(P, t)} = \boxed{\vec{\nabla} \cdot (\lambda(P) \vec{\nabla} T(P, t))} + \boxed{\frac{Q_v(t)}{V_{\text{source}}} f(P)}$$

Transient
Advection
Conduction
Internal source



Temperature rise at **any** point P:

$$\theta(P, t) = T(P, t) - T_{init}(P)$$

Its Laplace transform :

$$\bar{\theta}(P, p) = \int_0^{\infty} \exp(-p t) \theta(P, t) dt$$

Laplace parameter

Assumptions : Transient heat equation + boundary conditions with time-invariant coefficient + uniform initial temperature (the system is **Linear** and also **Time-Invariant** LITI)

$$\boxed{\rho c(P) \frac{\partial T}{\partial t}(P, t)} + \boxed{\rho c(P) \vec{u}(P) \cdot \vec{\nabla} T(P, t)} = \boxed{\vec{\nabla} \cdot (\lambda(P) \vec{\nabla} T(P, t))} + \boxed{\frac{Q_v(t)}{V_{source}} f(P)}$$

Transient
Advection
Conduction
Internal source

Consequences : Laplace transformed heat equation (**no time derivative**)

$$\boxed{\rho c(P) p \bar{\theta}(P, p)} + \boxed{\rho c(P) \vec{u}(P) \cdot \vec{\nabla} \bar{\theta}(P, p)} = \boxed{\vec{\nabla} \cdot (\lambda(P) \vec{\nabla} \bar{\theta}(P, p))} + \boxed{\frac{\bar{Q}_v(p)}{V_{source}} f(P)}$$

Transient
Advection
Conduction
Internal source

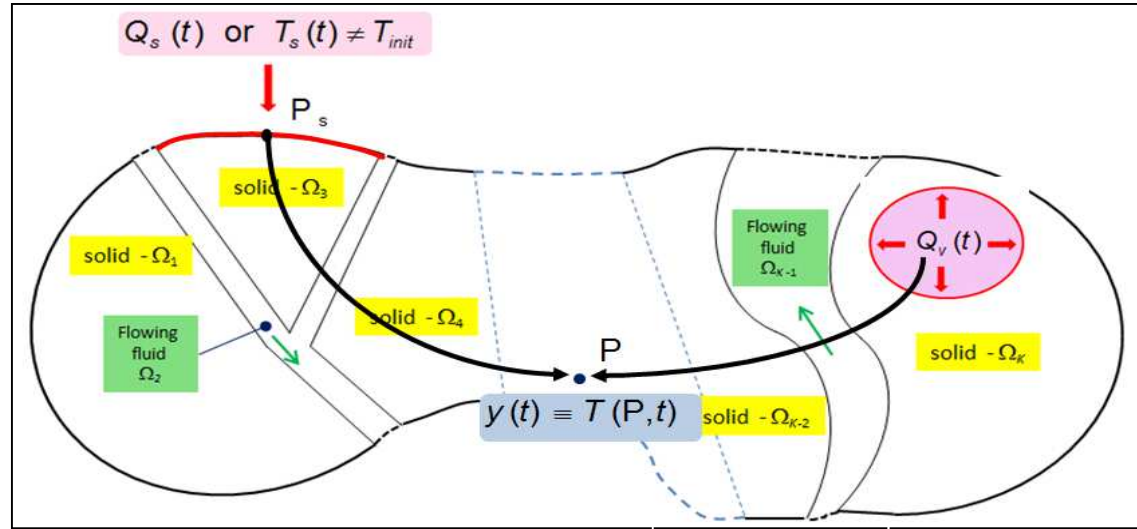
Linear system with a single excitation

Temperature or flux
 ⇒ response at any point P
 in the system

= simple product
 (Laplace domain)

$$\bar{y}(P, p) = \bar{H}(P, p) \bar{u}(p)$$

or convolution product (time domain)



response ←

excitation →

$$y(P, t) = H(P, t) * u(t) = \int_0^t H(P, t-t') u(t') dt'$$

Excitation $u(t)$:

$$u(t) = Q_v(t) - Q_v^{init} \text{ or } Q_s(t) - Q_s^{init}$$

$$\text{or } T_s(t) - T_{init}(P_s)$$

$$\text{or } T_\infty(t) - T_\infty^{init}$$

$$\text{or } T_b^{in}(t) - T_b^{in,init}$$

Transfer function

$$H(P, t)$$

« init » = initial steady state
 or uniform temperature field

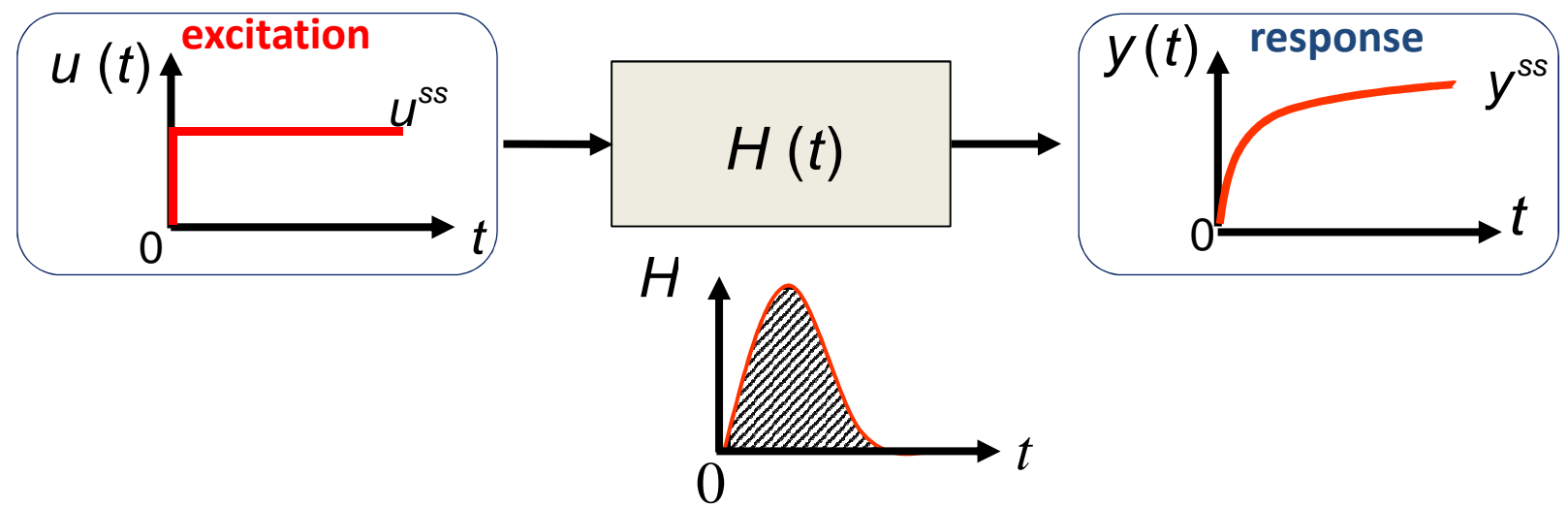
Response $y(t)$ in any specific point P :

$$y(t) = \theta(P, t) = T(P, t) - T_{init}(P)$$

or local heat flux in
 any direction $\varphi_x(P, t)$

Excitation u	Response y	Transfer function H
Power source Q (watts)	Temperature difference θ (kelvins)	Impedance Z (K.J ⁻¹)
Temperature difference θ (kelvins)	Temperature difference θ (kelvins)	Transmittance W (s ⁻¹)
Power source Q (watts)	Rate of heat flow Φ (watts)	Transmittance W (s ⁻¹)
Temperature difference θ (kelvins)	Rate of heat flow Φ (watts)	Admittance Y (W.K ⁻¹ .s ⁻¹) ⁹

$$y(P, t) = H(P, t) * u(t) = \int_0^t H(P, t-t') u(t') dt'$$



Steady state (ss) version of a transfer function

$$H^{ss} = \frac{y^{ss}}{u^{ss}} = \int_0^{\infty} H(t) dt$$

asymptotic values

Time distribution

case:

- $u \equiv Q$ or $Q - Q^{ss}$ (thermal power)
- $y \equiv \theta = T - T_{init}$ (P) (temperature variation)
- $\Rightarrow H \equiv Z$ (thermal impedance)

$$T_{in} - T_{out} = R \phi$$

Thermal resistance, flux pipe between 2 isothermal surfaces

$$T^{ss} - T_{\infty} = Z^{ss} Q$$

Generalized resistance, no flux pipe



2. The calibration problem

2.1 Case of a heat exchanger

Assumptions

- Constant thermo-physical properties (fluid and walls) and velocities (LTI heat equation) :

$$\partial \beta / \partial t = 0$$

$$\beta \equiv u_{hot}, u_{cold}, \lambda, \rho, \dots$$

- Uniform initial conditions/initial steady state)

$$T(P, t=0) = T_{init}$$

$$T(P, t=0) = T_{init}^{ss}(P)$$

- Heat losses through convection/(linearized) radiation with environment through a uniform heat transfer coefficient h at temperature $T_{\infty} = T_{init}$

- One **single heat source** (inlet temperature increase) that starts at $t = 0^+$:

$$\theta_1(t) = T_1(t) - T_{init} \neq 0$$

Cause

$$\theta_4(t) = T_4(t) - T_{init} = 0$$

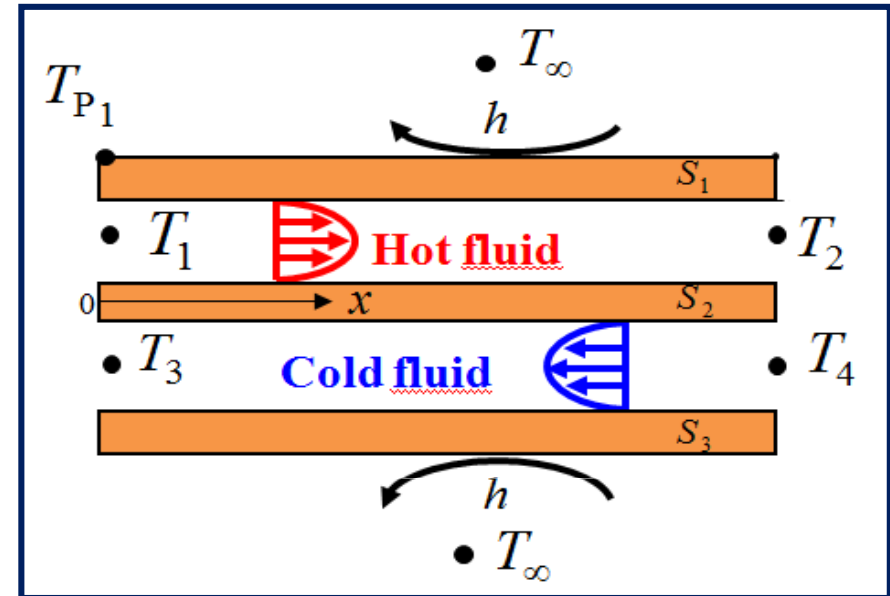


- Transient/unsteady thermal regime with **observed responses** at **any** point q :

$$\theta_q(t) = T_q(t) - T_{init}$$

$$\theta_q(t \leq 0) = 0 \quad \text{and} \quad \theta_q(t > 0) \neq 0$$

Consequences



Calculation of convolution products (transmittance case)
 Parameterizing with piecewise constant functions, square case

response transmittance unique pseudo source

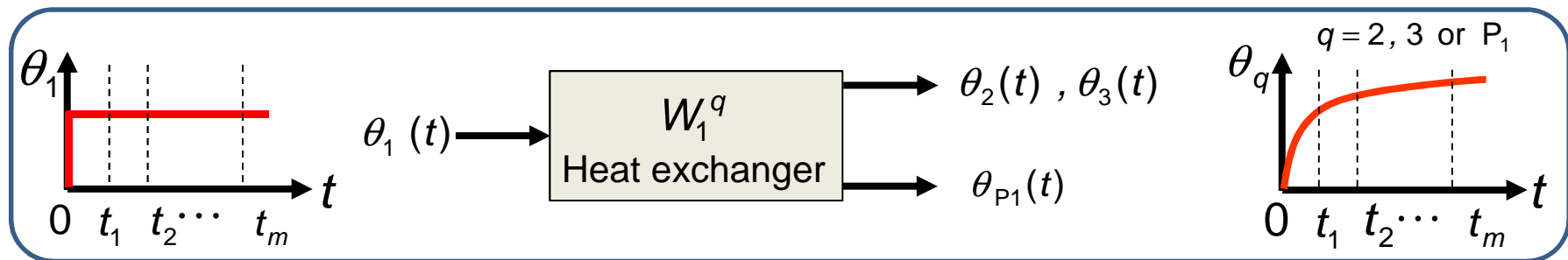
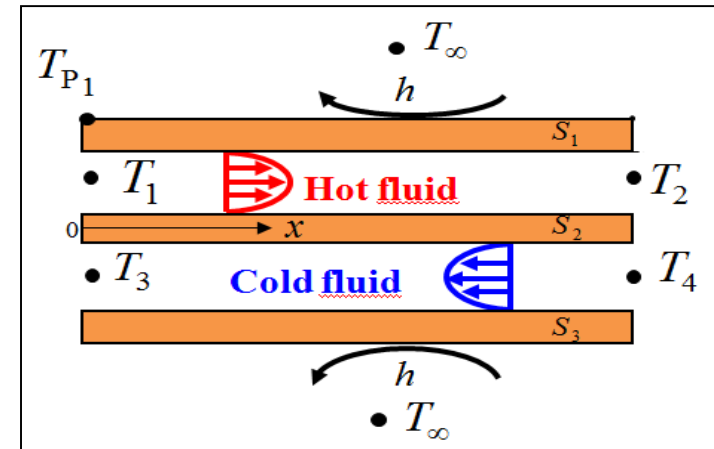
$$\begin{aligned} \theta(P, t) &= W(P, t) * \theta_1(t) \\ &= \int_0^t W(P, t-t') \theta_1(t') dt' \\ &= \int_0^t \theta_1(P, t-t') W(P, t') dt' \end{aligned}$$

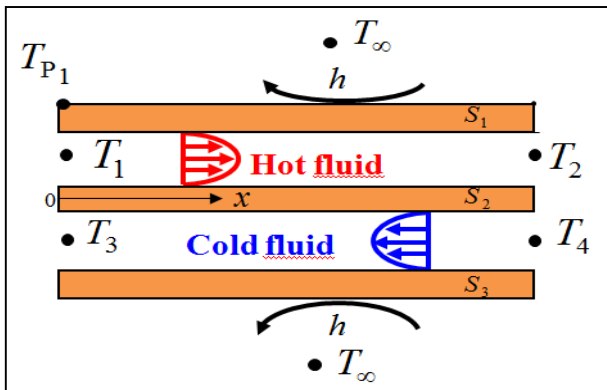
$$\theta(P, t_i) \approx \Delta t \sum_{j=1}^m \theta_{1,i-j+1} W_j(P)$$

sampled averaged over 1 time interval

$$t_0 = 0 ; t_i = i \Delta t \text{ for } i = 1 \text{ to } m ; \Delta t = t_{final} / m$$

$$z_i = \frac{1}{\Delta t} \int_{t_{i-1}}^{t_i} z(t) dt \approx \frac{1}{2} (z(t_{i-1}) + z(t_i)) \text{ for } z(t) = \theta_1 \text{ or } W(P)$$





$$\mathbf{M}(z) \equiv \Delta t \begin{bmatrix} z_1 & & & & \\ z_2 & z_1 & & & 0 \\ z_3 & z_2 & z_1 & & \\ \vdots & \vdots & \vdots & \ddots & \\ z_m & z_{m-1} & z_{m-2} & \cdots & z_1 \end{bmatrix}$$

$\mathbf{M}(z)$ Lower triangular Toeplitz matrix function of a vector z

$$z_i = \frac{1}{\Delta t} \int_{t_{i-1}}^{t_i} z(t) dt \approx \frac{1}{2} (z(t_{i-1}) + z(t_i)) \text{ for } z(t) = \theta_1 \text{ or } W(P)$$

$$\theta_q = \begin{pmatrix} \theta_q(t_1) \\ \theta_q(t_2) \\ \vdots \\ \theta_q(t_m) \end{pmatrix}, q = 2, 3, 4 \text{ or } P_1$$

instantaneous (sampled) values

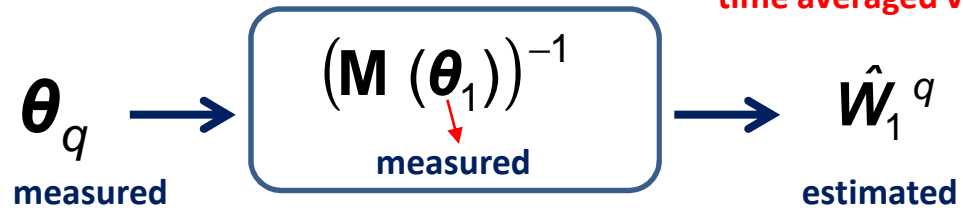
$$\theta_q = \mathbf{M}(\theta_1) \quad W_1^q = \mathbf{M}(W_1^q) \theta_1$$

$$w = \begin{pmatrix} w_1 \\ w_2 \\ \vdots \\ w_{1m} \end{pmatrix} \quad \theta_1 = \begin{pmatrix} \theta_{1,1} \\ \theta_{1,2} \\ \vdots \\ \theta_{1,m} \end{pmatrix}$$

time averaged values

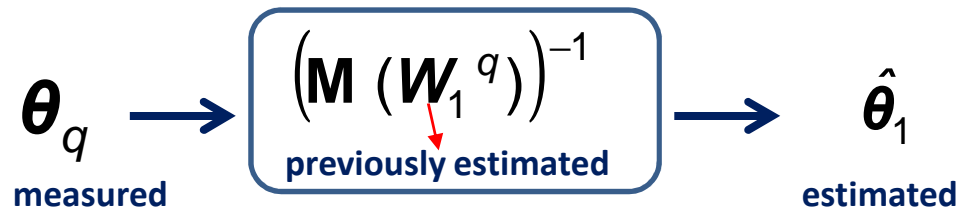
➤ **First experiment:**

- Calibration (inverse) problem



➤ **Next experiments:**

- virtual sensor inverse problem (same as source estimation problem)



Model for W identification calibration problem:

$$\theta_q = \mathbf{M}(\theta_1) W^q$$

exact
unknown
↓
↓

$$\theta_q^{exp} = \theta_q + \varepsilon_q \quad \text{and} \quad \theta_1^{exp} = \theta_1 + \varepsilon_1$$

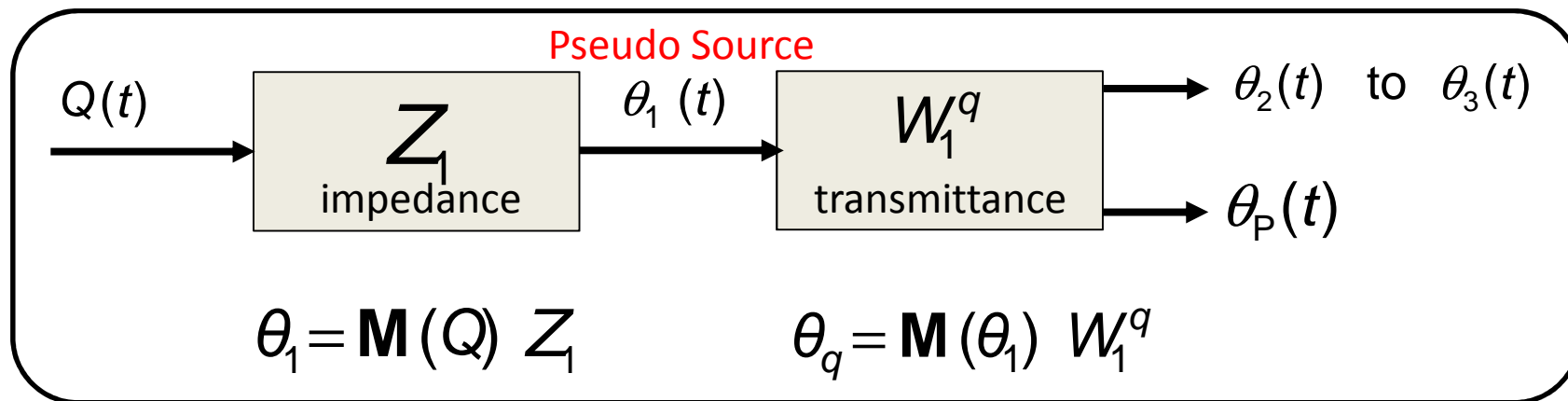
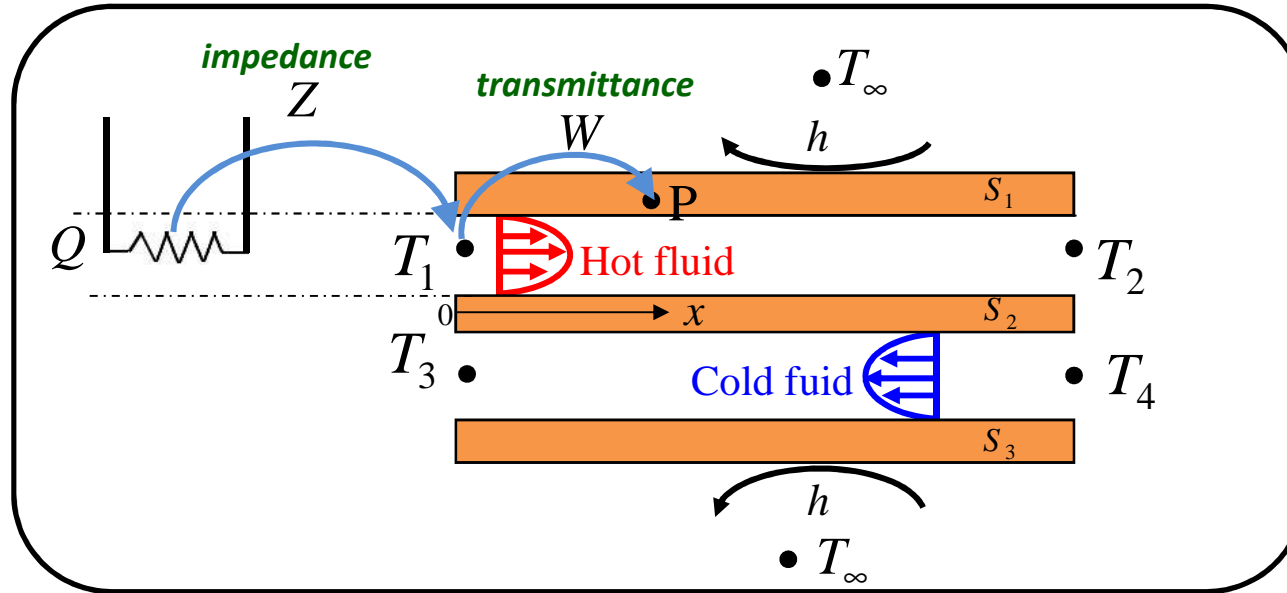
measured
↙
↘
↙
↘
 2 i.i.d. and independent noises

- Ordinary (linear) least squares: $\hat{W}^q = \left(\mathbf{M}(\theta_1^{exp}) \right)^{-1} \theta_q^{exp}$

Ill-posed problem:
 Inversion needs **regularization**
 Here: Truncated SVD or 0 order Tikhonov

- with discrepancy principle (Morozov)

Practical way of making the inlet temperature vary

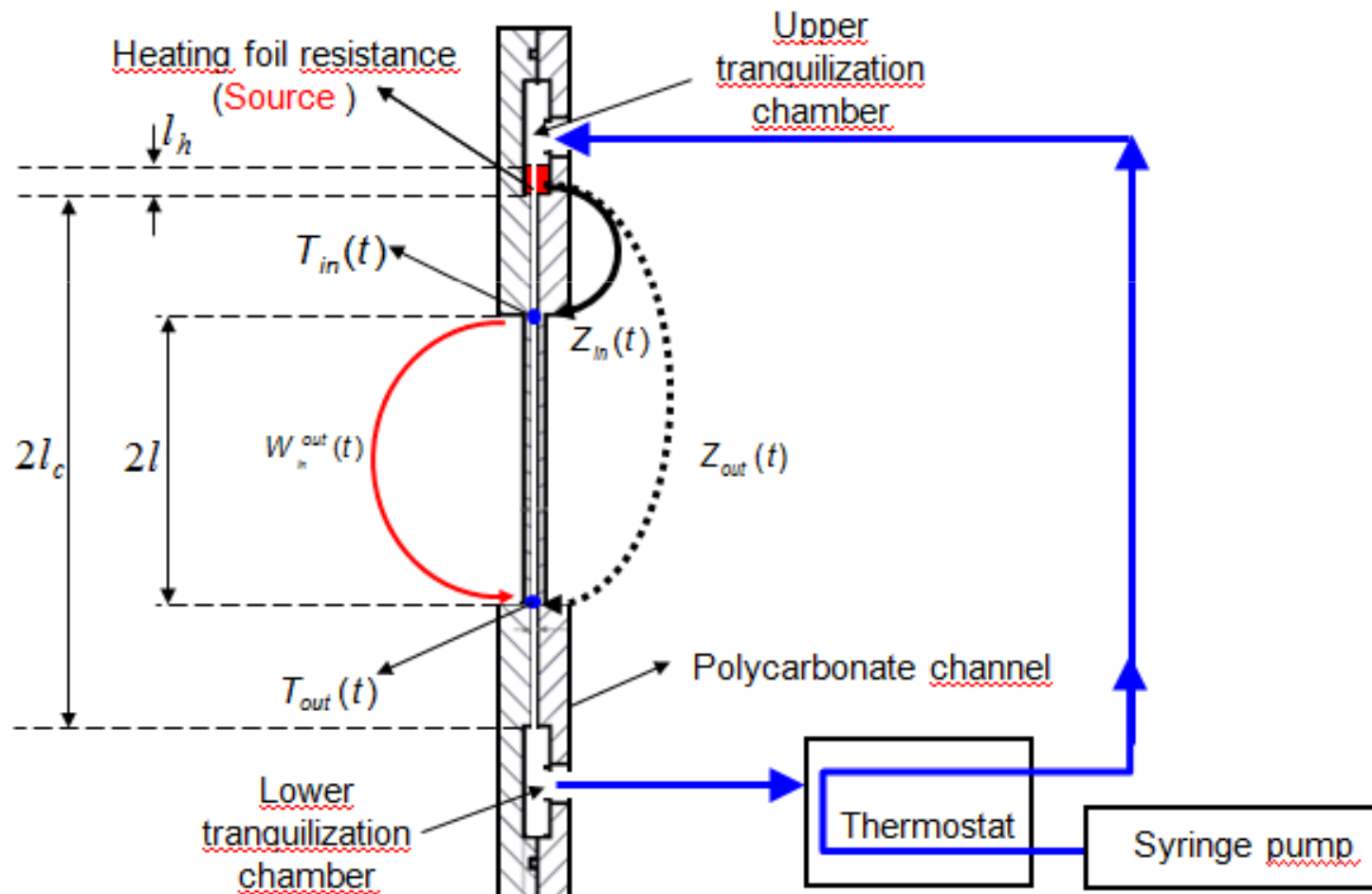


2.2 – Experimental Impedance/transmittance estimation for a half heat exchanger

Identification of transmittance using experimental transient measurements (calibration)

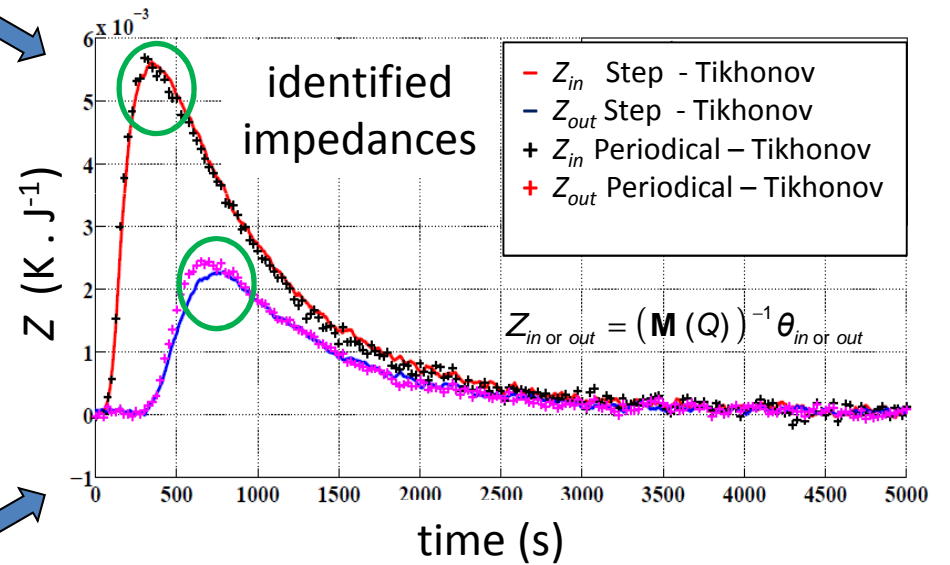
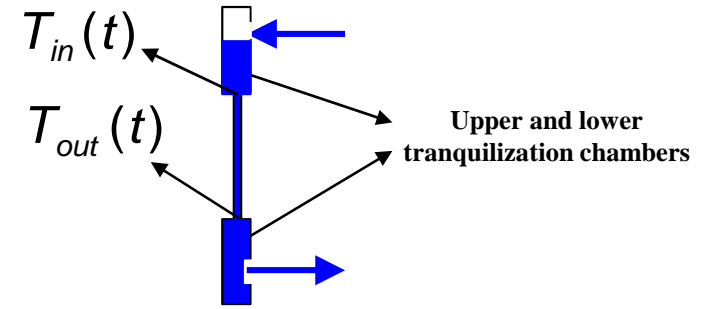
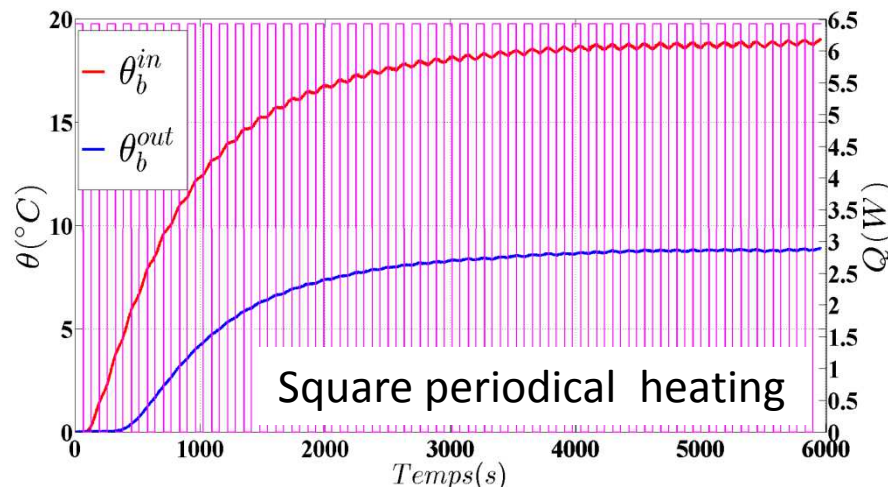
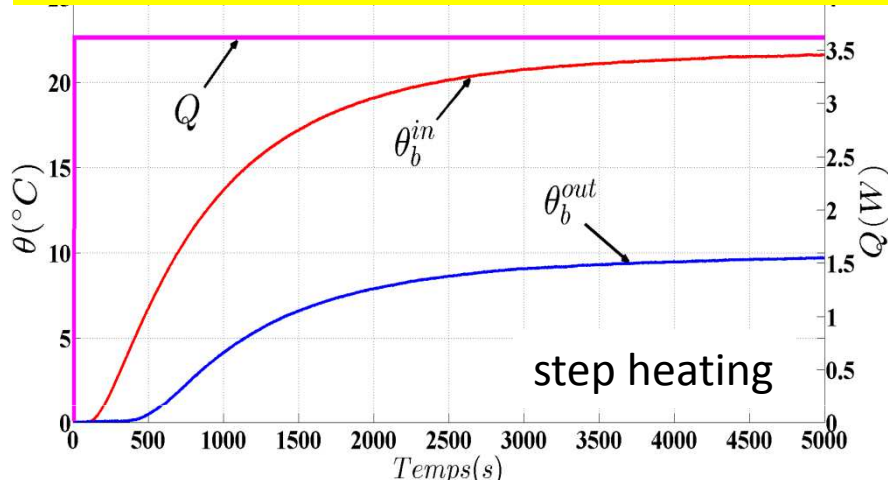
Response : thermocouples
 $d = 50.8 \mu\text{m}$

$e_1 = e_2(\text{mm})$	$e_f(\text{mm})$	$2l(\text{mm})$	$2l_c(\text{mm})$	$w(\text{mm})$	$l_h(\text{mm})$
2	1	65	120	50	10

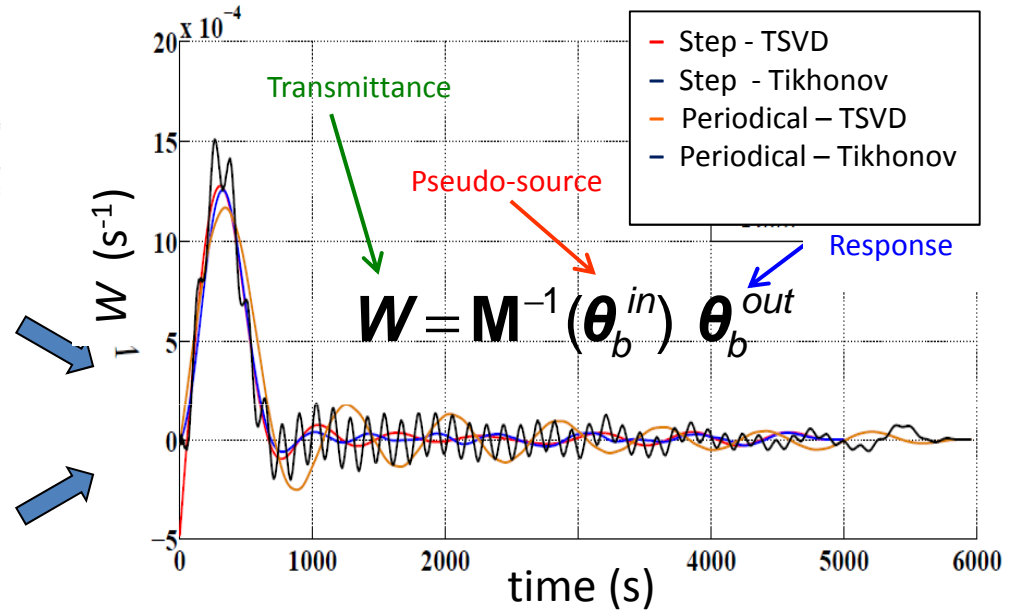
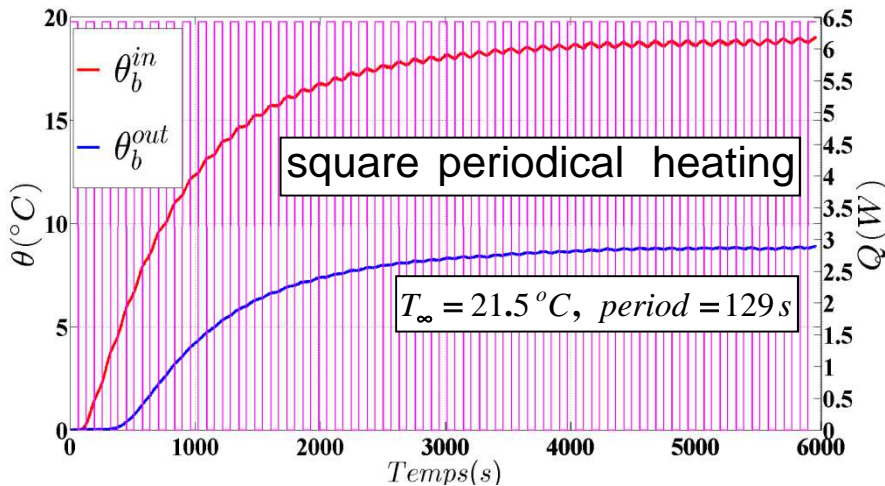
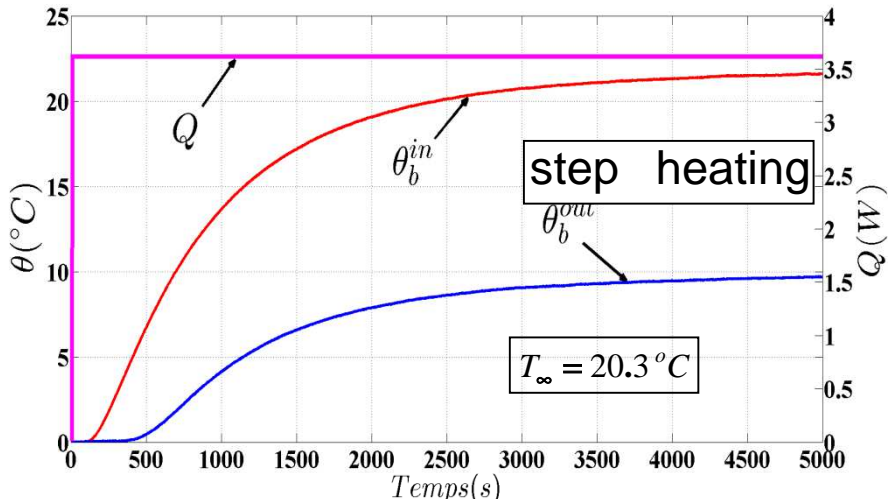


Identification of transfer function using experimental temperature recording:

Comparison of identified Z : step or periodical heating



Comparison of identified transmittance W (outlet/inlet):
step or periodical heating

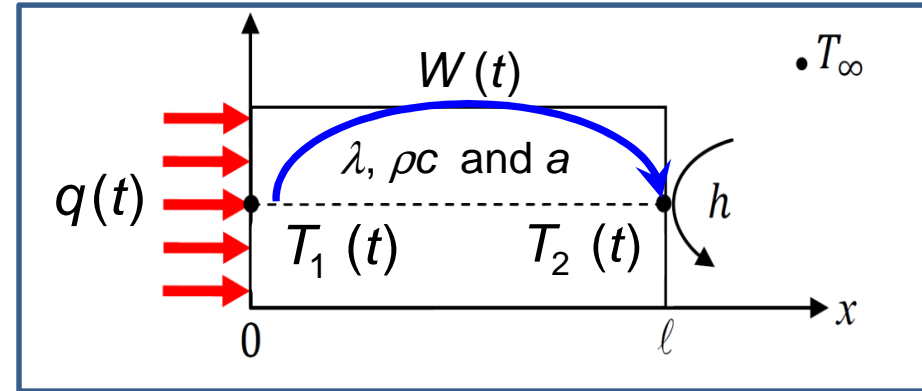


Oscillations past first peak and for long times,
zero initial level hard to recover :

Estimation of transmittance W (noisy output
and input) more difficult than Z (noisy output only)

3. Analysis of deconvolution deadlocks

3.1 Reference case: 1D transient conduction



Heat eq. $\left\{ \begin{array}{l} \frac{\partial^2 \theta}{\partial x^2} = \frac{1}{a} \frac{\partial \theta}{\partial t} \end{array} \right.$

boundary $\left\{ \begin{array}{l} \varphi = -\lambda \frac{\partial \theta}{\partial x} = q(t) \quad \text{at } x = 0 \quad \text{for } t > 0; \\ \varphi = -\lambda \frac{\partial \theta}{\partial x} = h \theta \quad \text{at } x = l \quad \text{for } t > 0 \end{array} \right.$

$$\theta(t) = T(t) - T_{\infty}$$

change of variables

initial $\left\{ \theta = 0 \quad \text{at } t = 0 \quad \text{for } 0 \leq x \leq l \right.$

$$H \equiv W, \quad u \equiv \theta_1, \quad y \equiv \theta_2$$

Laplace transform

$$\bar{\psi}(x, p) \equiv \int_0^t \psi(x, t) \exp(-pt) dt \quad \text{for } \psi = \theta \text{ or } \varphi$$

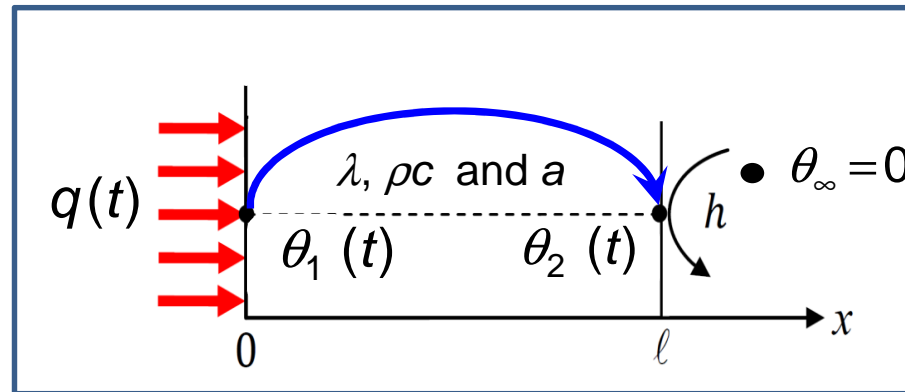
$$\bar{\theta}_2(p) = \bar{W}(p) \bar{\theta}_1(p) \quad \Leftrightarrow \quad \theta_2(t) = W(t) * \theta_1(t)$$

Transmittance

$$W(t) = L^{-1} \left[\frac{1}{\cosh(\beta l) + h \sinh(\beta l) / (\lambda \beta)} \right]$$

with $\beta^2 = p/a$

Comparison: analytical W and identified W from synthetic profiles (COMSOL)



Analytical Laplace W + numerical inversion of Laplace

$$W(t) = L^{-1} \left[\frac{1}{\cosh(\beta l) + h \sinh(\beta l) / (\lambda \beta)} \right]$$

versus

Identified W from θ_1 and θ_2

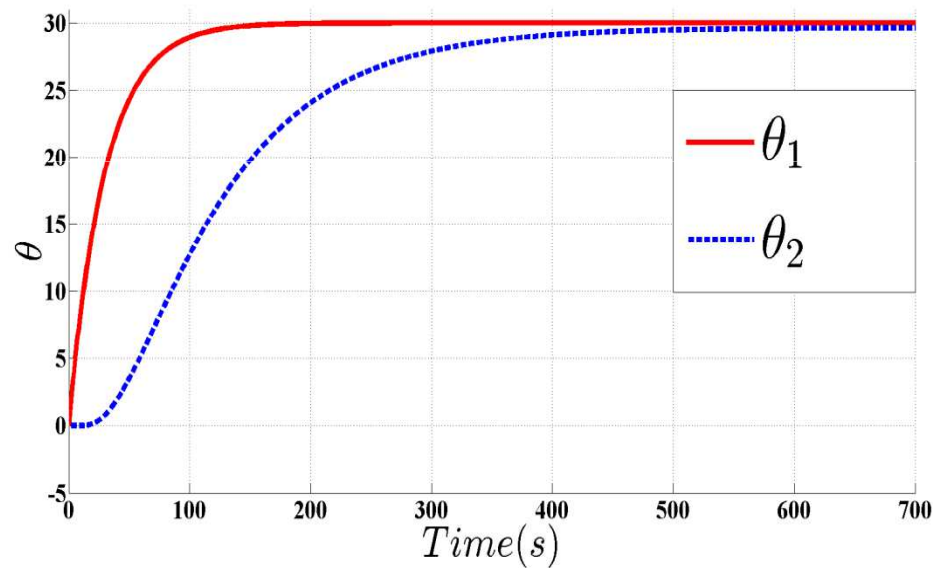
Numerical Inversion of Laplace Transforms by Hoog's algorithm

Input : $\theta_1(t) = \left(1 - e^{-\frac{t}{\tau}} \right) \theta_1^{ss}$ with $\tau = 30 \text{ s}$; $\theta_1^{ss} = 30 \text{ }^\circ\text{C}$ and $\Delta t = 0.5 \text{ s}$

t_f (s)	l (mm)	h (W.m ⁻² .K ⁻¹)	λ (W.m ⁻¹ .K ⁻¹)	ρc (kJ.m ⁻³ .K ⁻¹)
700	50	10	43	3666

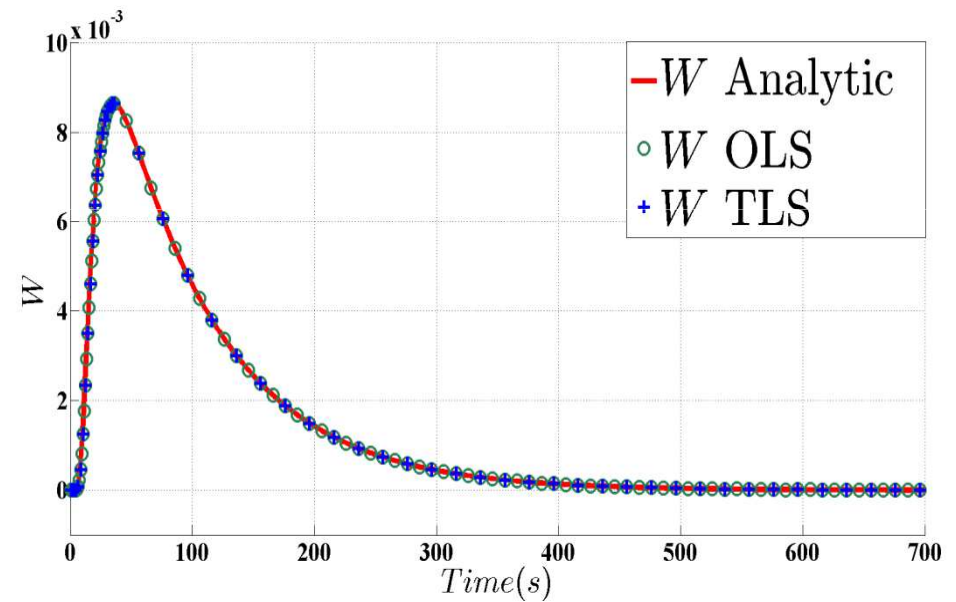
Comparison: analytical W and identified W from synthetic profiles (COMSOL)

Comparison without noise:



Temperature profiles (COMSOL)

Validation without committing an INVERSE CRIME



Analytical and identified W
(without regularization)

3.2 Noisy matrix and Total Least Squares

Calibration problem: how to get the best transfer function $H(t)$ at a point P?

Model

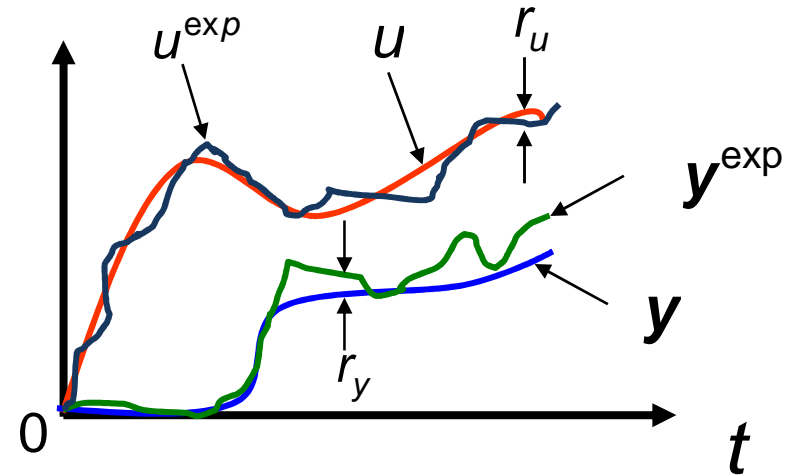
$$y(P, t) = u(t) * H(P, t) = \int_0^t u(t-t') H(t-t') dt'$$

↓ sampling ↓ parameterization

$$\mathbf{y} = \mathbf{M}(u) \mathbf{H} \quad \text{or} \quad \mathbf{y} = \mathbf{A} \mathbf{H} \quad \text{with} \quad \mathbf{A} = \mathbf{M}(u)$$

Measurements

Available information:
discrete noisy values
of $y(P, t)$ and $u(t)$



$$y^{exp}(t_i) = y(t_i) + \varepsilon_i$$

↑ noise

$$u^{exp}(t_i) = u(t_i) + \tau_i$$

$$\Rightarrow \mathbf{A}^{exp} = \mathbf{A} + \boldsymbol{\varepsilon}_u \quad \text{with} \quad \boldsymbol{\varepsilon}_u = \mathbf{M}(\boldsymbol{\tau})$$

↑ noisy matrix

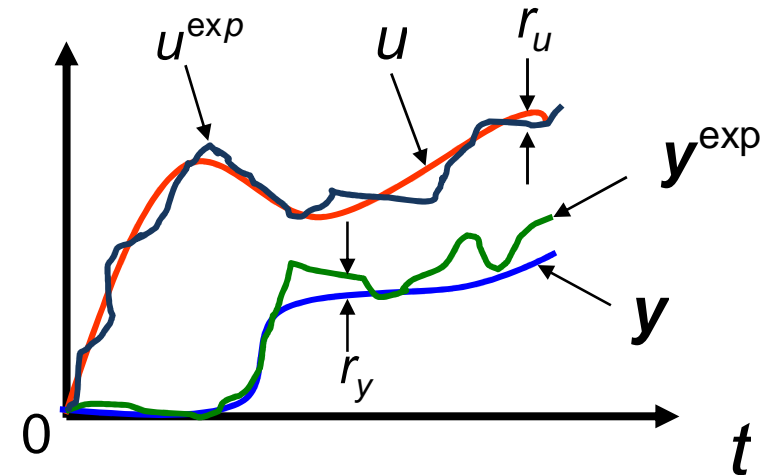
H Identification from u and y measurements : $y = M(u) H$

Total Least Squares (TLS) solution :

augmented matrix
$$G = \left[\begin{array}{c|c} M(u) & y \end{array} \right]$$

\uparrow \uparrow
 concatenation

\hat{H} such as minimum residuals r_y and r_u



Frobenius norm:
$$J_{TLS}(H) = \|r_G(H)\|_F^2 = \sum_{i=1}^m \sum_{j=1}^{m+1} g_{ij}^2$$

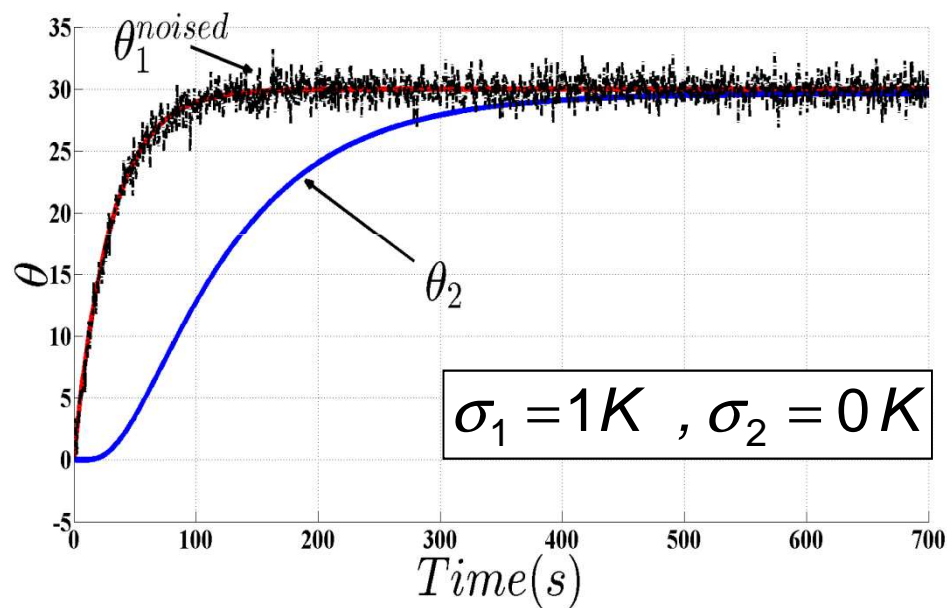
with
$$r_G(H) = G^{exp} - G$$

SVD Form of G \longrightarrow **ill-posed** \longrightarrow Regularized form: Truncated TLS

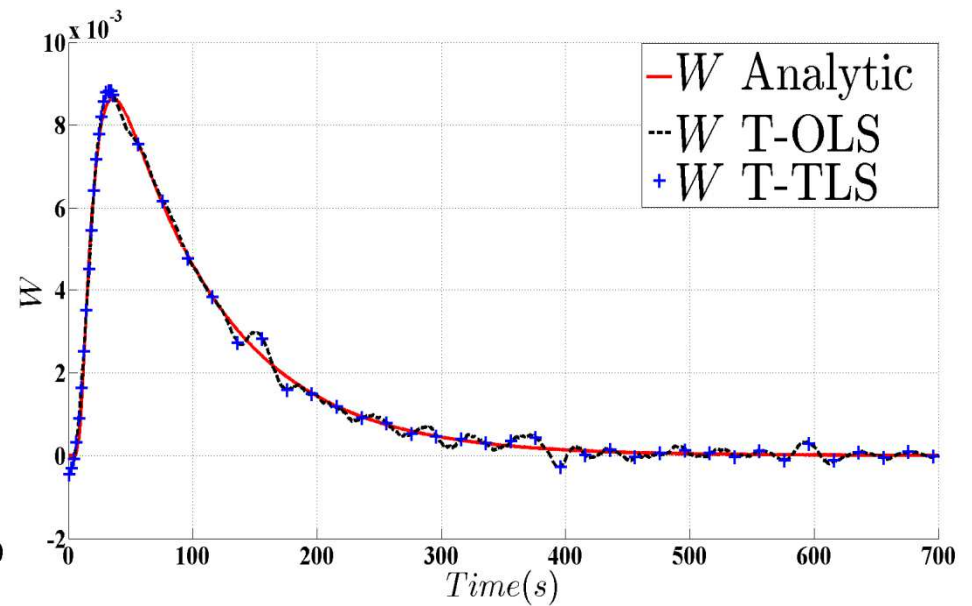
3.3 Comparison of calibration methods

Comparison: analytical W and identified W from synthetic profiles (COMSOL)

Comparison with noise: θ_1 noised



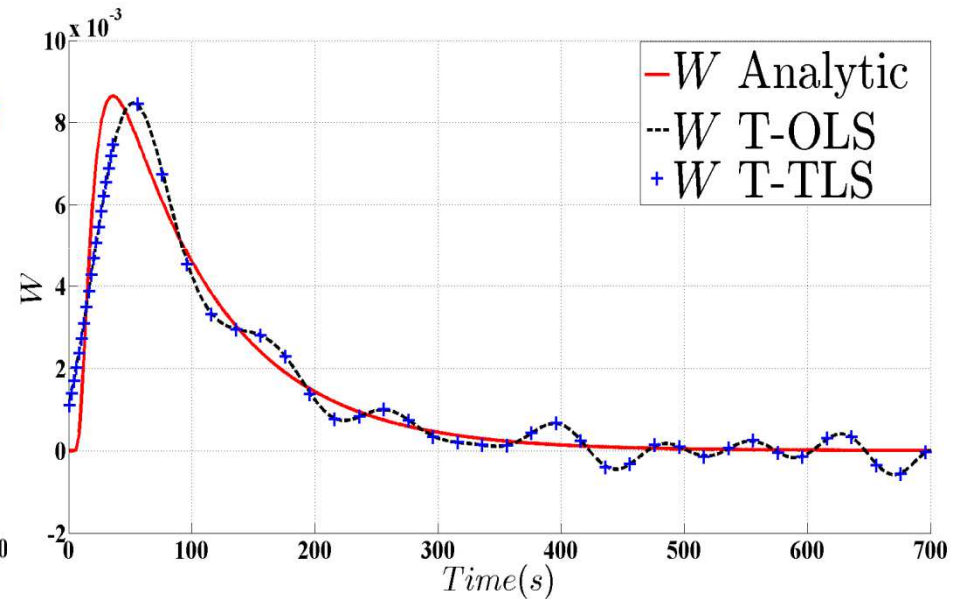
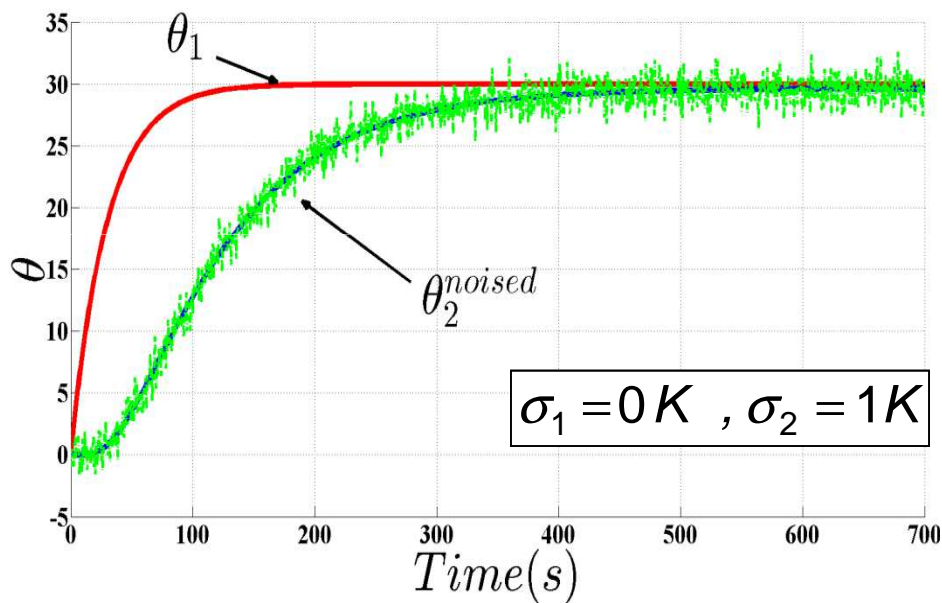
Temperature profiles (COMSOL)



Analytical and identified W
(with regularization)

Comparison: analytical W and identified W from synthetic profiles (COMSOL)

Comparison with noise: θ_2 noised



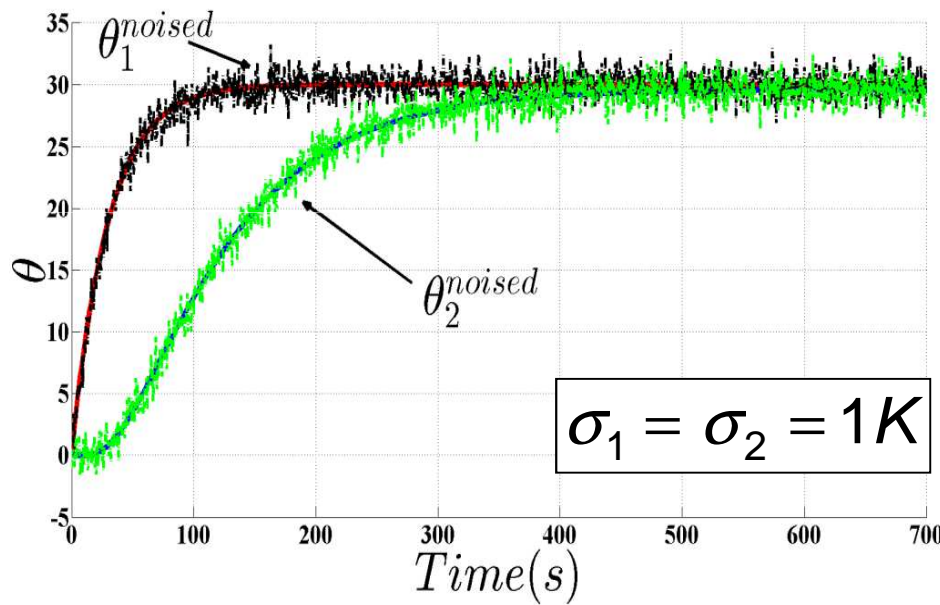
Temperature profiles (COMSOL)

Analytical and identified W
(with regularization)

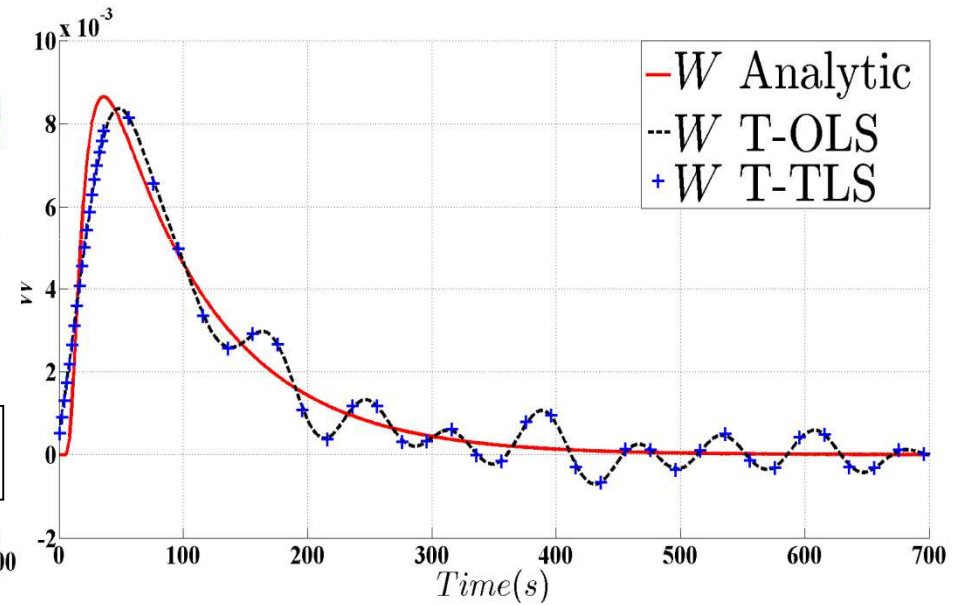
Noise over signal ratio: $\| \theta_2^{noised} - \theta_2^{exact} \| / \| \theta_2^{noised} \| = 3.90 \%$

Comparison: analytical W and identified W from synthetic profiles (COMSOL)

Comparison with noise: θ_1, θ_2 noised

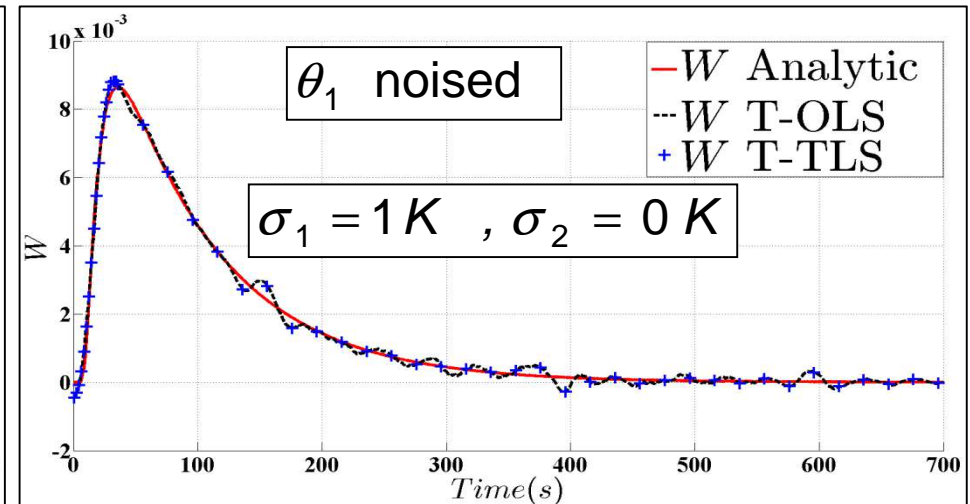
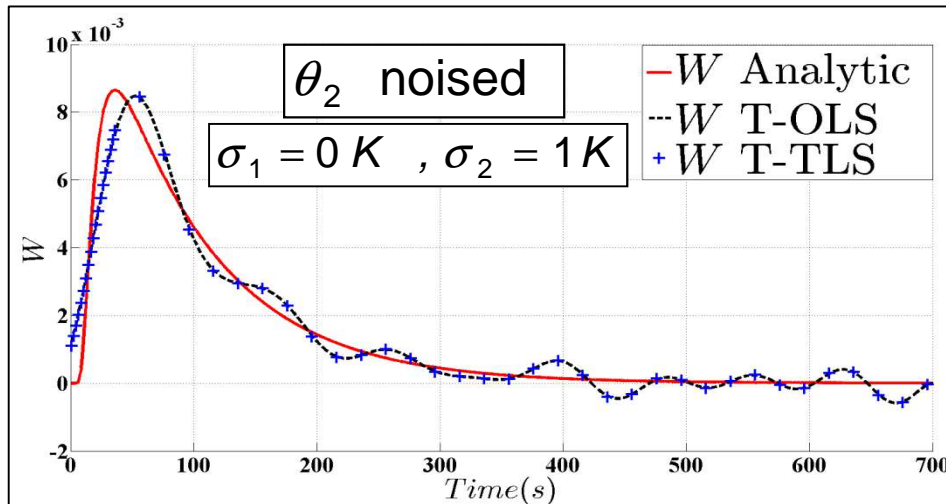


Temperature profiles (COMSOL)

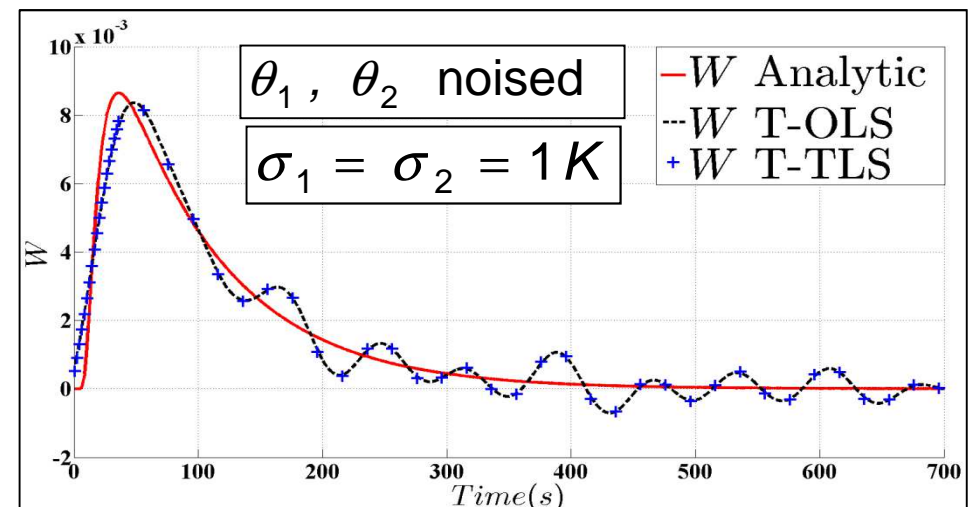


Analytical and identified W
(with regularization)

Conclusion of this comparison of deconvolution techniques



- Noise on the response θ_2 more penalizing than noise on the source θ_1 .
- The truncated total least squares do not take into account the convolutive structure of the matrix $M(\theta_1)$
 \Rightarrow no improvement of the estimate
- Important to improve short times values of the identified TF in a calibration experiment:
 \rightarrow largest impact on posterior inverse input estimation experiments.

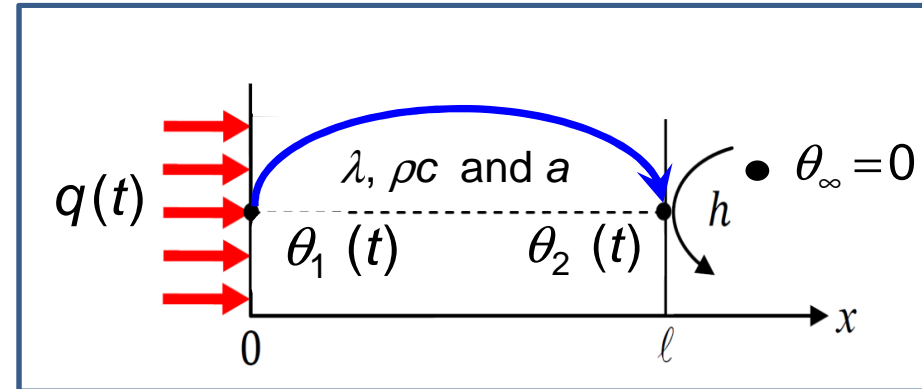


4. Rectangular deconvolution

Calibration problem :

$$\theta_2 = \mathbf{A} \mathbf{W} \quad \text{with} \quad \mathbf{A} = \mathbf{M}(\theta_1)$$

\downarrow \downarrow \downarrow
 m output $n=m$ unknowns m input
 data data data

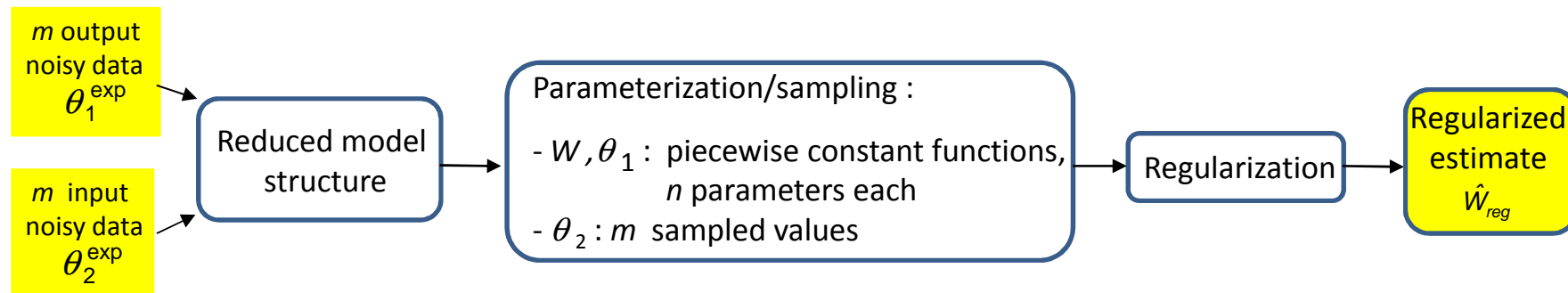


Can the ill-posed problem be more **parsimonious** ?

→ Less many unknowns n than output data: $n \ll m$

Calibration = **model identification** for a given structure

Idea: tailoring the definition of the estimate



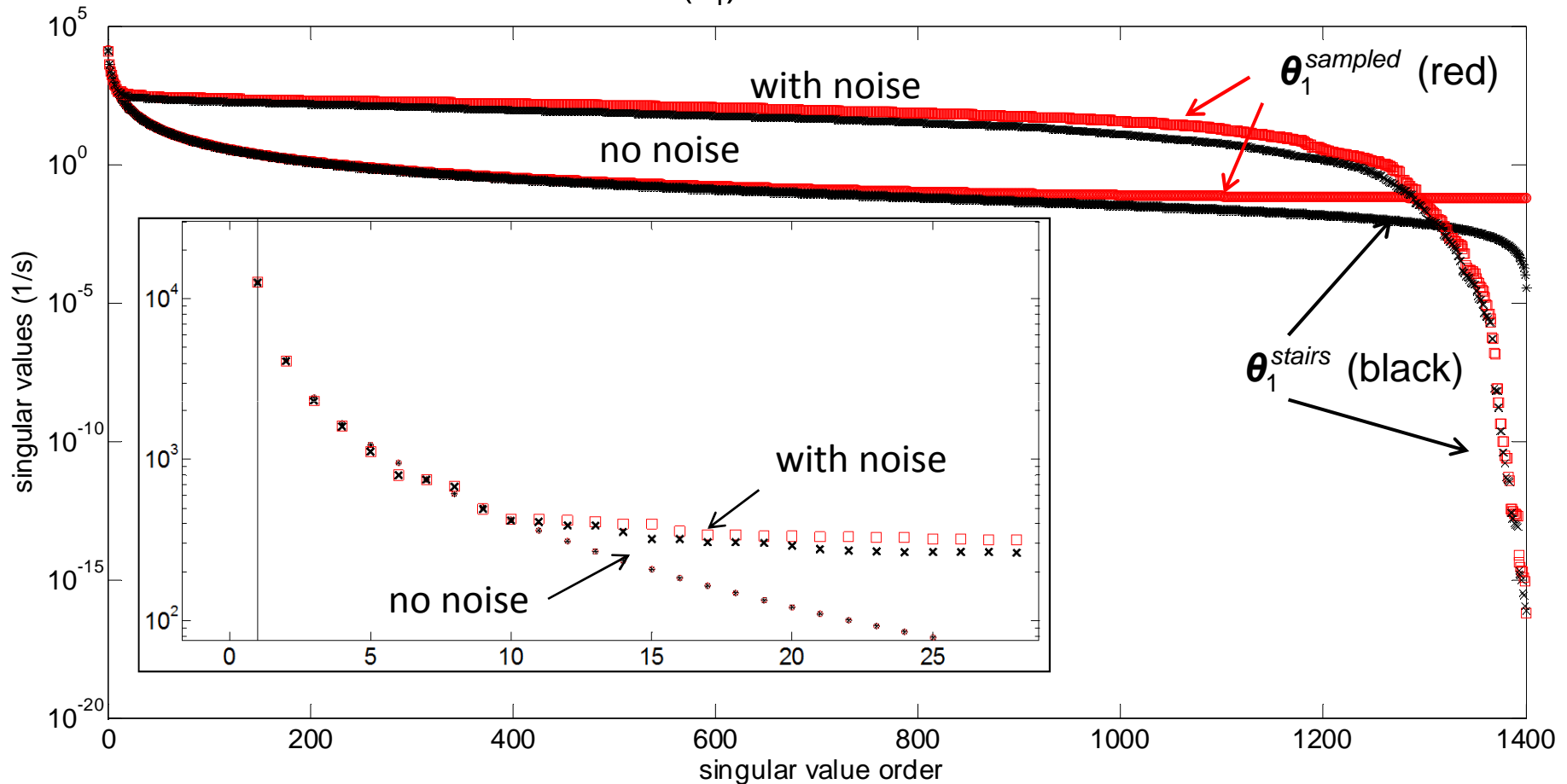
2 paths deserve to be investigated **BEFORE** regularization:

- Rectangular parametrization using piecewise constant parametrization for W
- ARX model construction (perspective)

4.1. Point versus averaged values for input and unknown

Singular values of $\mathbf{M}(\theta_1)$ depending on type of parameterization of θ_1 and on its noise

$$\mathbf{M}(\theta_1) = \mathbf{U} \mathbf{S} \mathbf{V}^T$$



$$s_1 \left(\mathbf{M}(\theta_{1,\alpha}^{\text{stairs},\sigma_1}) \right) \approx s_1 \left(\mathbf{M}(\theta_{1,\alpha}^{\text{sampled},\sigma_1}) \right)$$

$$\text{cond} \left(\mathbf{M}(\theta_{1,\alpha}) \right) = s_1 / s_\alpha \quad \text{for } 1 \leq \alpha \leq m$$

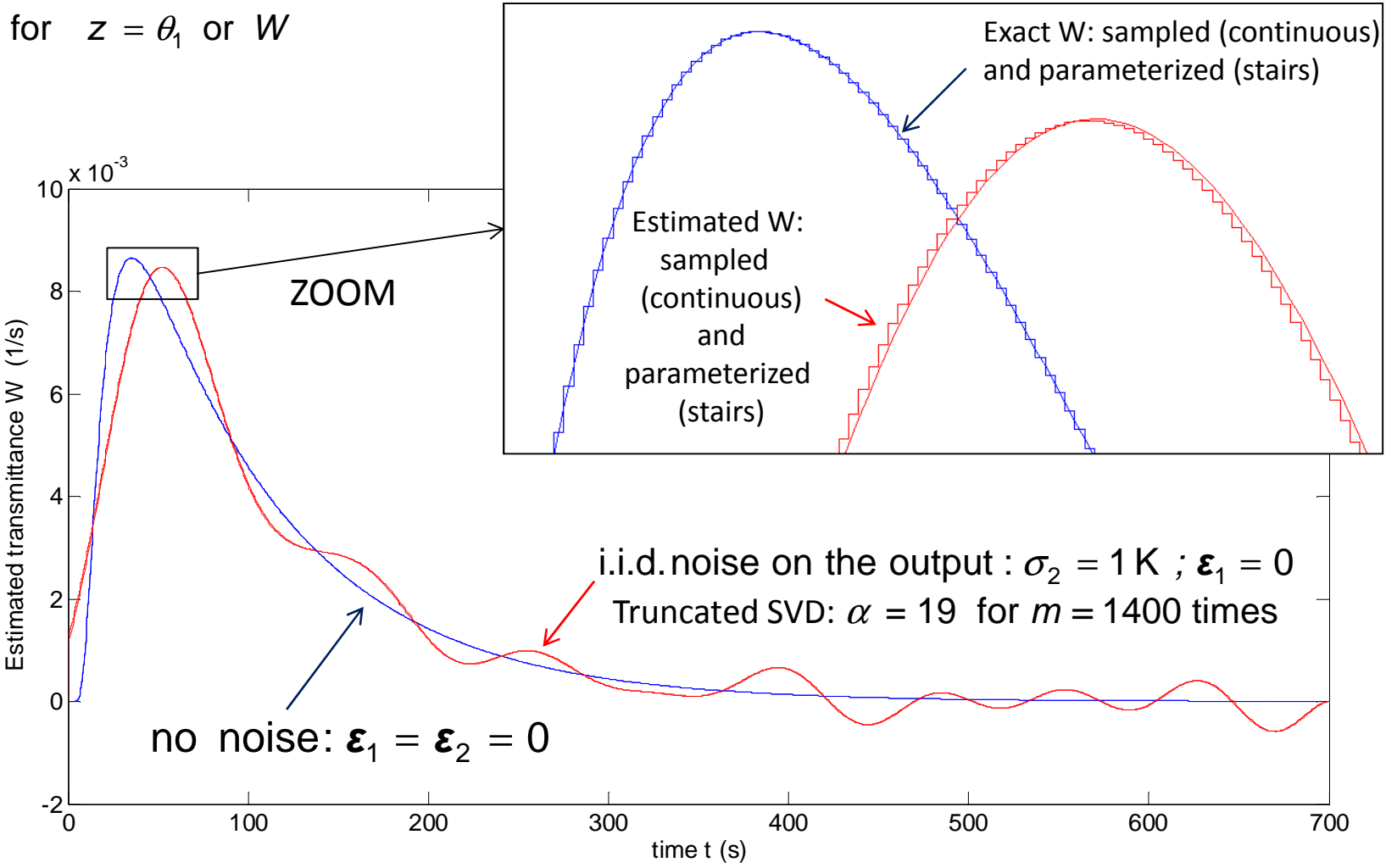
$$\alpha : \text{truncation order} - \sigma_1 = \text{noise level} \quad \text{cond} \left(\mathbf{M}(\theta_{1,\alpha}^{\text{stairs},\sigma_1}) \right) \geq \text{cond} \left(\mathbf{M}(\theta_{1,\alpha}^{\text{sampled},\sigma_1}) \right)$$

Effect of the type of parameterization: noise on θ_2 only

Sampled = parameters = instantaneous value (parameterization over a basis of « hat » functions) : $z_i = z(t_i)$

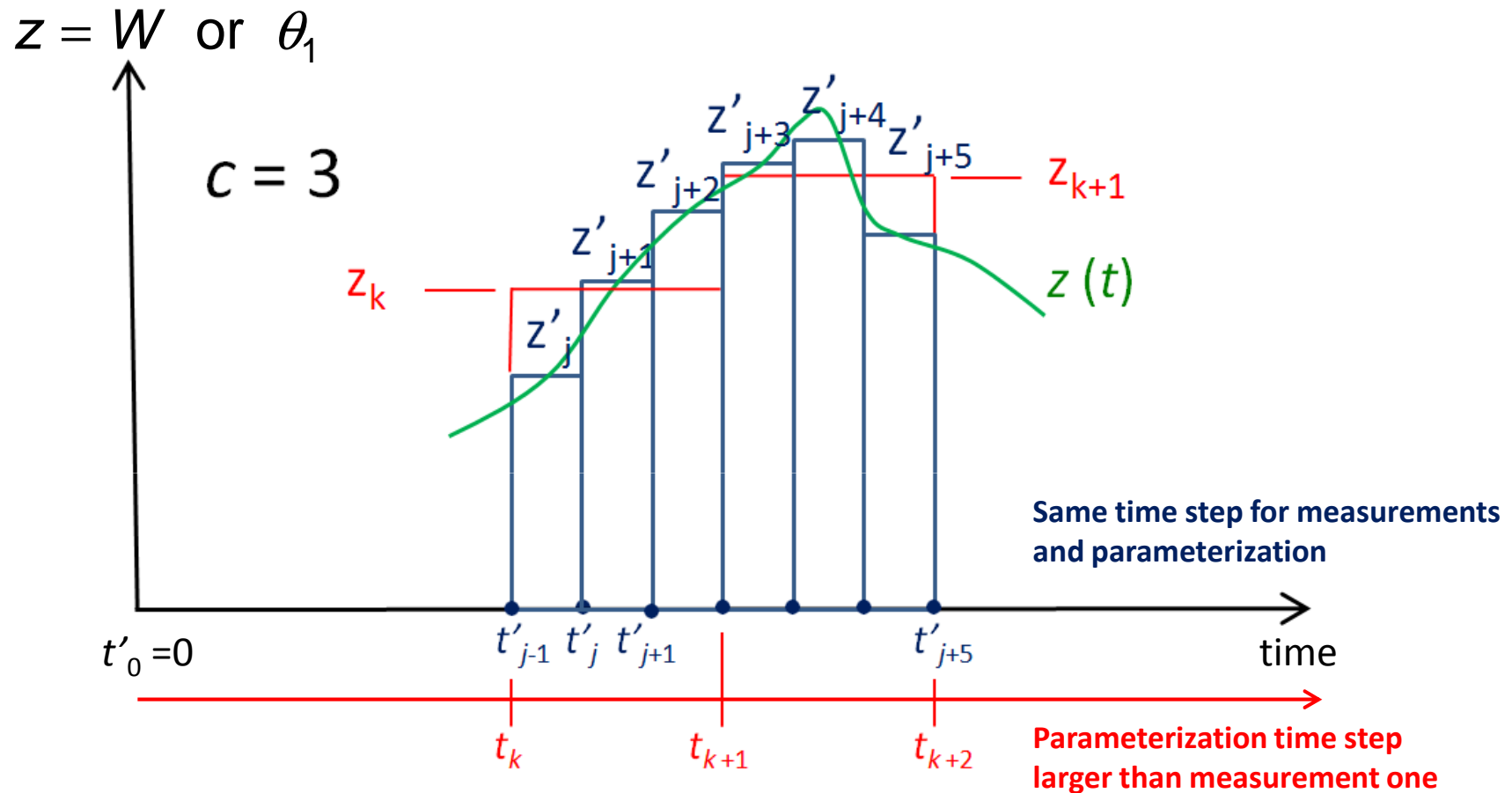
Parameterized = averaged value (parameterization over a base of « doors » functions) : $z_i = 0.5 (z(t_i) + z(t_{i-1}))$

for $z = \theta_1$ or W



$$\text{bias}(\beta) = E(\hat{\beta}) - \beta^{\text{exact}} \qquad \left\| \text{bias} \left(\hat{W}_{\alpha_{opt}}^{\text{stairs}, \sigma_2} \right) \right\| \leq \left\| \text{bias} \left(\hat{W}_{\alpha_{opt}}^{\text{sampled}, \sigma_2} \right) \right\|$$

4.2 Rectangular deconvolution (using « stairs » parameterizing)



➤ Less many parameters than measurement times: $n \ll m$

➤ Simplest method = use of a basis of n piecewise constant functions :
 $m/n = c$ (integer)

Square model :

$$\theta_2^{square} = \mathbf{M}(\theta_1) \mathbf{W}^{square} = \mathbf{A} \mathbf{W}^{square}$$

$m \times 1$ $m \times m$ $m \times 1$ $m \times m$

$$\mathbf{A} = \begin{bmatrix} \mathbf{A}_1 & \mathbf{A}_2 & \dots & \mathbf{A}_n \end{bmatrix}$$

sensitivity matrix sensitivity vectors

Rectangular model :

$$\theta_2^{rectangular} = \mathbf{M}(\theta_1) \mathbf{W}^{square} = \mathbf{X} \mathbf{W}$$

$m \times 1$ $m \times n$ $m \times 1$ $m \times n$ $n \times 1$

repetition of some lines

$$\mathbf{W}^{square} = \mathbf{G} \mathbf{W}$$

$m \times 1$ $m \times n$ $n \times 1$

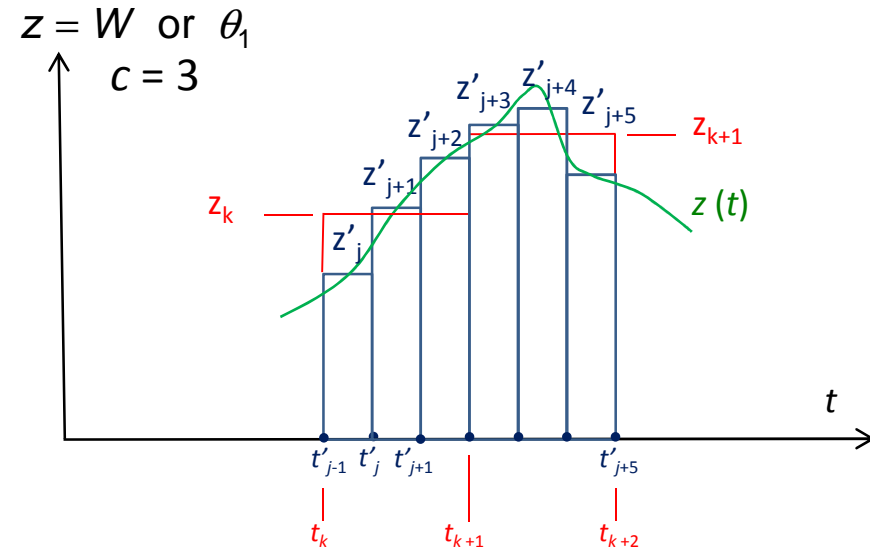
local averaging

$$\theta_2^{rectangular} = \mathbf{A} \mathbf{G} \mathbf{W} = \mathbf{X} \mathbf{W}$$

$m \times 1$ $m \times n$ $m \times n$ $n \times 1$

$$\mathbf{X} = \begin{bmatrix} \mathbf{X}_1 & \mathbf{X}_2 & \dots & \mathbf{X}_n \end{bmatrix}$$

sensitivity matrix sensitivity vectors



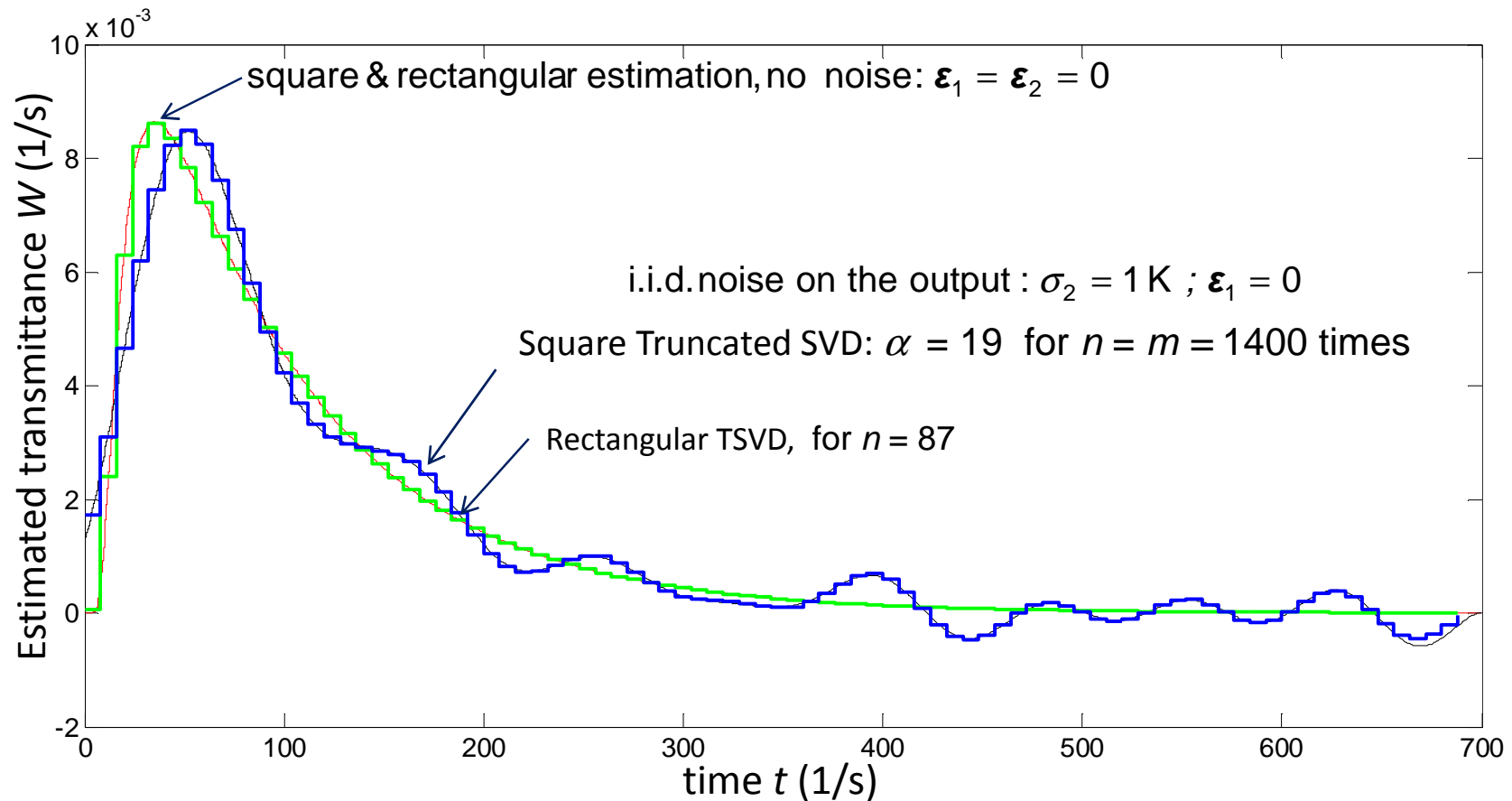
$$\mathbf{X}_k = \frac{1}{C} \sum_{j=(k-1)c+1}^{kc} \mathbf{A}_j$$

The n rectangular sensitivity vectors are simply the averaged values of the m square sones

$$\mathbf{A}_j = \Delta t$$

$$\begin{bmatrix} 0 \\ \vdots \\ 0 \\ \theta_{1,j} \\ \theta_{1,j+1} \\ \vdots \\ \theta_{1,m-j+1} \end{bmatrix}$$

Effect of rectangular inversion: 1 unknown every $c = 16$ time steps



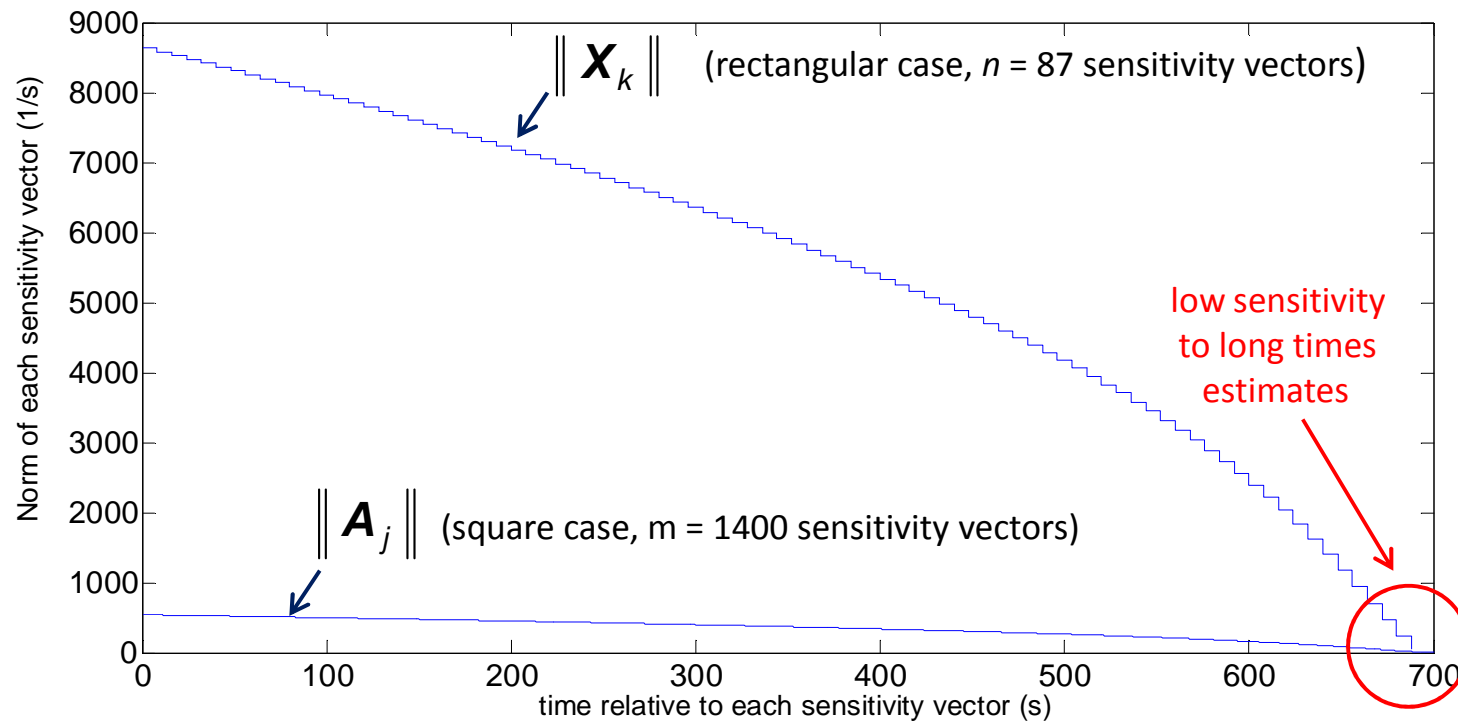
Rectangular case with $n = 87$ unknowns $\text{cond}(\mathbf{A}) = 2530$

Same relative RMS residual $\left\| \boldsymbol{\theta}_2^{\sigma_2} - \boldsymbol{\theta}_2^{\text{square, recalculated}}(\hat{\mathbf{W}}_{\alpha_{\text{opt}}}) \right\| / \left\| \boldsymbol{\theta}_2^{\sigma_1} \right\| = 3.88 \%$

$\left\| \boldsymbol{\theta}_2^{\sigma_2} - \boldsymbol{\theta}_2^{\text{rectangular, recalculated}}(\hat{\mathbf{W}}_{\text{OLS}}) \right\| / \left\| \boldsymbol{\theta}_2^{\sigma_1} \right\| = 3.88 \%$

No gain in term of estimation bias

Study of the norm (length) of the sensitivity vectors



4.3 Rectangular estimation with $n < m$ non uniform (NU) time steps

$$\mathbf{x}_k^{NU} = \frac{1}{c_k} \sum_{a_{k-1}+1}^{a_k} \mathbf{A}_j \quad \text{where} \quad a_k = \sum_{k'=1}^{k-1} c_{k'}$$

Different number of elementary time steps

Each of the n rectangular sensitivity vectors are simply the averaged values of c_k square ones

$$\mathbf{A}_j = \Delta t \begin{bmatrix} 0 \\ \vdots \\ 0 \\ \theta_{1,j} \\ \theta_{1,j+1} \\ \vdots \\ \theta_{1,m-j+1} \end{bmatrix}$$

$$\theta_2^{rectangular, NU} \underset{m \times 1}{=} \mathbf{X}^{NU} \underset{n \times 1}{\mathbf{W}} \quad \text{where} \quad \mathbf{X}^{NU} \equiv [\mathbf{x}_1 \quad \mathbf{x}_2 \quad \dots \quad \mathbf{x}_n]$$

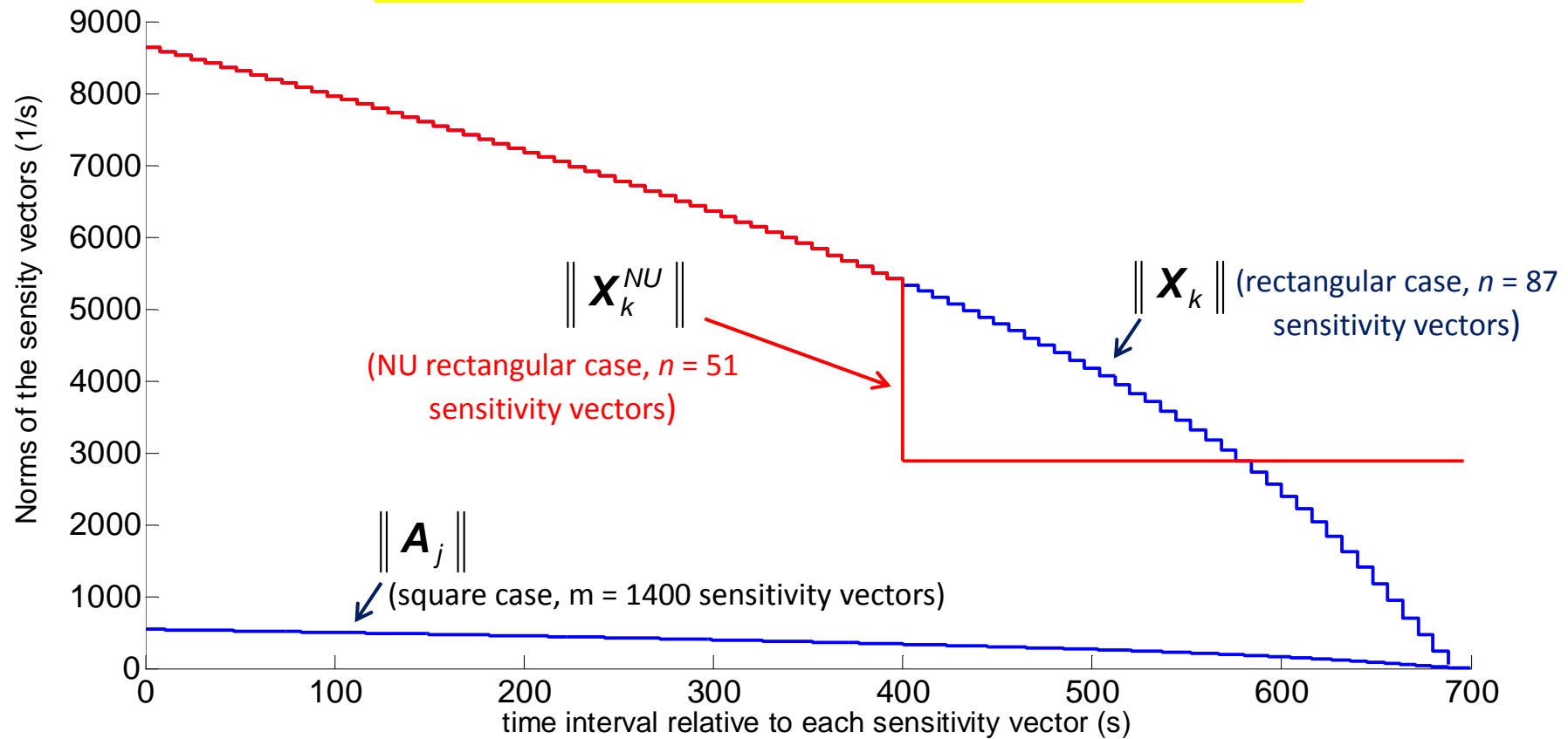
$$\text{with } \mathbf{X}^{NU} \underset{m \times n}{=} \mathbf{G}^{NU} \underset{m \times m}{\mathbf{A}} = \mathbf{G}^{NU} \underset{\substack{\text{local} \\ \text{averaging}}}{\mathbf{M}(\theta_1)}$$

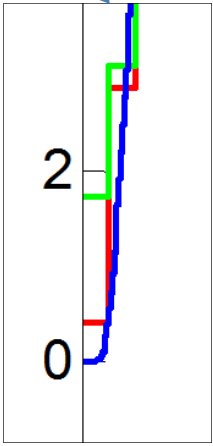
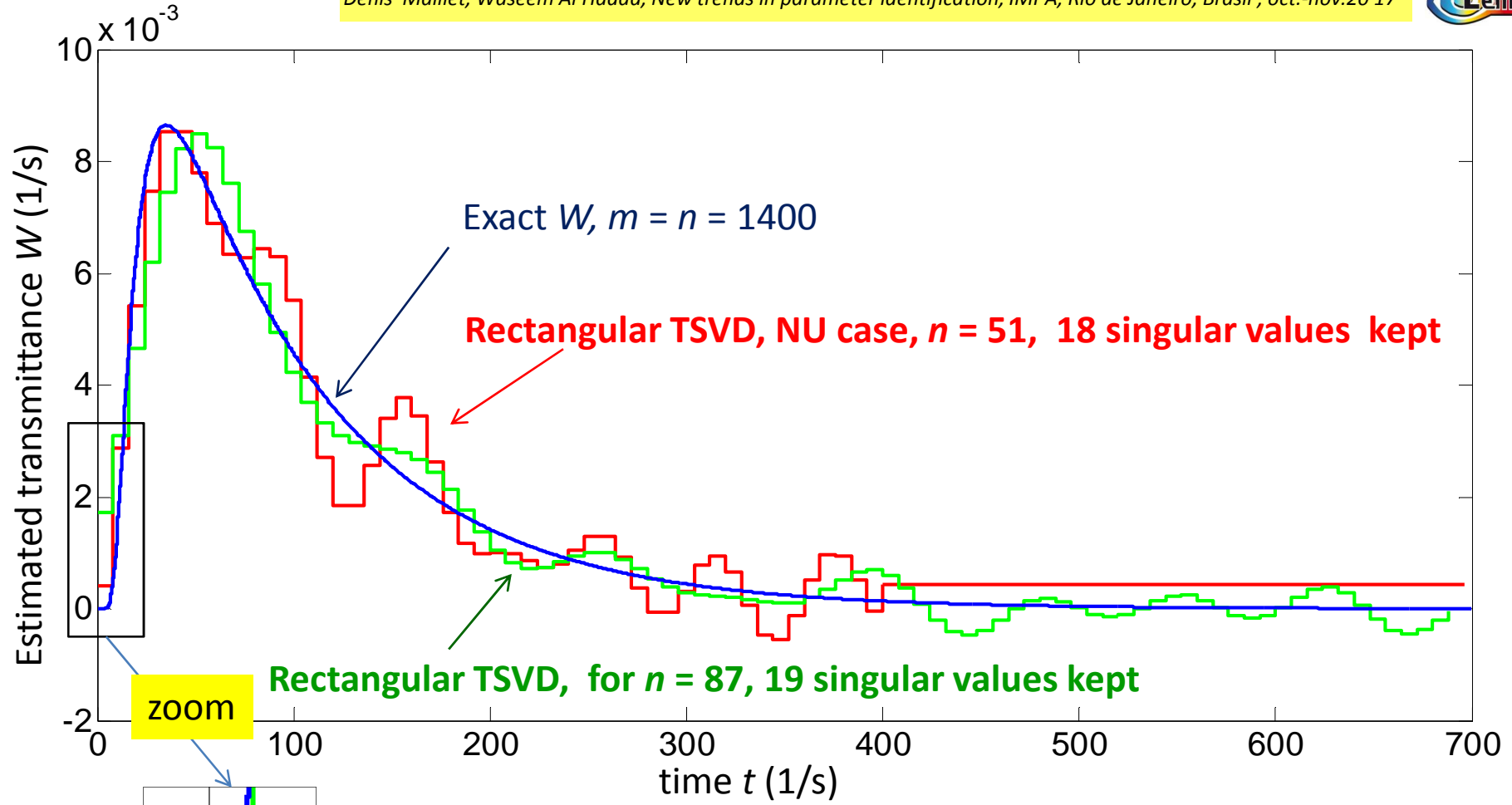
lower triangular Toeplitz matrix (square)

Question: how to chose the n limits of the n different time steps ?

First try: constant level past time $t = 400$ s (steady state reached)

Study of the norm (length) of the sensitivity vectors





Improvement of zero level and of the transmittance for short times

5. Conclusions/perspectives

- Importance and applicability of **transfer functions** (impedances, transmittances, ...) in (exact) **reduced** convolutive model structures for **Linear Time Invariant** physical systems (detailed model = PDE, integro-differential equations, ...)

- Convolution products can be given a **commutative** vector/matrix form in discrete time
 - ⇒ ill-posed inverse problems: identification problem (**calibration**) **first**, inverse input problem (**source estimation**) or inverse (or direct) **virtual sensor**, or **use in a Non Destructive Testing** procedure **next**

- Path to improve the quality of estimation of transmittance: **rectangular deconvolution** and pertinent way of **tayloring** its unknown parameters

- Perspective: use of **ARX structures** (AutoRegressive models with eXternal inputs) for better estimation of transfer functions

Muito obrigado!

Model for W identification calibration problem:

$$\theta_q = \mathbf{M}(\theta_1) \mathbf{W}^q$$

exact unknown

$$\theta_q^{exp} = \theta_q + \epsilon_q \quad \text{and} \quad \theta_1^{exp} = \theta_1 + \epsilon_1$$

measured

2 i.i.d. and independent noises

- Ordinary least squares: $\hat{\mathbf{W}}^q = \left(\mathbf{M}(\theta_1^{exp}) \right)^{-1} \theta_q^{exp}$

Ill-posed problem:
Inversion needs **regularization**
Here: Truncated SVD or 0 order Tikhonov

- SVD decomposition of **square** sensitivity matrix:

$$\mathbf{M}(\theta_1^{exp}) = \mathbf{U} \mathbf{S} \mathbf{V}^T \quad \text{with} \quad \mathbf{S} = \text{diag}(s_1 \quad s_2 \quad \dots \quad s_m)$$

singular values

-TSVD: $\hat{\mathbf{W}}_\alpha^q = \mathbf{V} \mathbf{S}_\alpha^{-1} \mathbf{U}^T \theta_q^{exp}$ with $\mathbf{S}_\alpha^{-1} = \text{diag}(1/s_1 \quad 1/s_2 \quad \dots \quad 1/s_\alpha \quad 0 \quad \dots \quad 0)$

- Zero order Tikhonov: $\hat{\mathbf{W}}_\mu^q = \text{Arg} \left(\min_{\mathbf{W}} \left(\underbrace{\| \mathbf{r}(\mathbf{W}) \|_2^2}_{\text{ordinary least squares sum}} + \mu \| \mathbf{W} \|_2^2 \right) \right)$ where $\mathbf{r}(\mathbf{W}) \equiv \theta_q^{exp} - \mathbf{M}(\theta_1^{exp}) \mathbf{W}$
residual vector

or: $\hat{\mathbf{W}}_\mu^q = \mathbf{V} \mathbf{S}_\mu^{-1} \mathbf{U}^T \theta_q^{exp}$ with $\mathbf{S}_\mu^{-1} = \mathbf{F}_\mu \mathbf{S}$ where $\mathbf{F}_\mu = \text{diag} \left(\frac{s_1^2}{\mu^2 + s_1^2} \quad \frac{s_2^2}{\mu^2 + s_2^2} \quad \dots \quad \frac{s_m^2}{\mu^2 + s_m^2} \right)$

- Choice of the hyperparameters $\gamma = \alpha$ or μ by **discrepancy principle** (Morozov) : $\| \mathbf{r}(\hat{\mathbf{W}}_\gamma) \|_2^2 \approx m \sigma^2$

standard deviation of ϵ_q 40

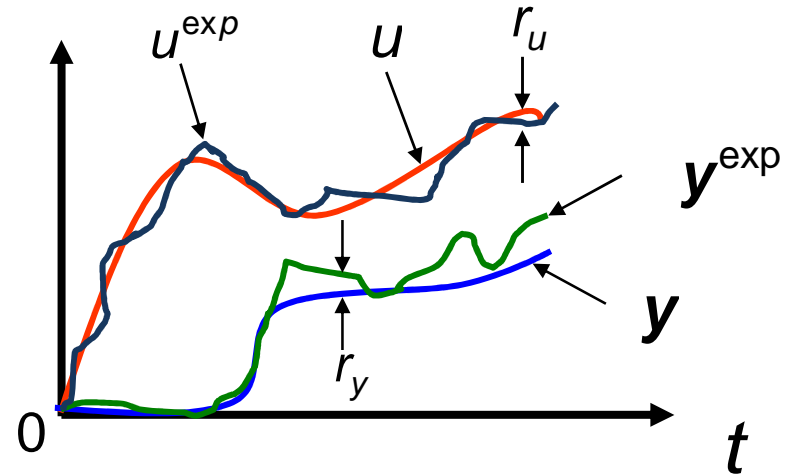
Total Least Squares (TLS) solution : augmented matrix $\mathbf{G} = \left[\mathbf{M}(u) \mid \mathbf{y} \right]$

$\hat{\mathbf{H}}$ such as minimum residuals r_y and r_u

concatenation

$$J_{TLS}(\mathbf{H}) = \|\mathbf{r}_G(\mathbf{H})\|_F^2 = \sum_{i=1}^m \sum_{j=1}^{m+1} g_{ij}^2$$

with $\mathbf{r}_G(\mathbf{H}) = \mathbf{G}^{\text{exp}} - \mathbf{G}$



SVD form

ill-posed

Regularized form:
T-TSVD

with $\hat{\mathbf{H}}_{TLS} = -\mathbf{V}_{12} \mathbf{V}_{22}^{-1}$
and $\mathbf{G}^{\text{exp}} = \mathbf{U} \mathbf{\Sigma} \mathbf{V}^T$
and $\mathbf{V} = \begin{bmatrix} \mathbf{V}_{11} & \mathbf{V}_{12} \\ \mathbf{V}_{21} & \mathbf{V}_{22} \end{bmatrix}$

$\hat{\mathbf{H}}_{T-TLS} = -\mathbf{V}_{12}^{\alpha_2} (\mathbf{V}_{22}^{\alpha_2})^T / \|\mathbf{V}_{22}^{\alpha_2}\|^2$
with $\mathbf{V} = \begin{bmatrix} \mathbf{V}_{11}^{\alpha_2} & \mathbf{V}_{12}^{\alpha_2} \\ \mathbf{V}_{21}^{\alpha_2} & \mathbf{V}_{22}^{\alpha_2} \end{bmatrix}$

4.1. Point versus averaged values for input and unknown

Back to the parameterization problem

$$\theta_2 = \mathbf{M}(\theta_1) \mathbf{W}$$

$$\mathbf{M}(\mathbf{z}) \equiv \Delta t \begin{bmatrix} z_1 & & & & \\ z_2 & z_1 & & & 0 \\ z_3 & z_2 & z_1 & & \\ \vdots & \vdots & \vdots & \ddots & \\ z_m & z_{m-1} & z_{m-2} & \cdots & z_1 \end{bmatrix}$$

- $z(t) = W(t)$ or $\theta_1(t)$: defined on a basis of m piecewise constant functions

$$z(t) = \sum_{j=1}^m z_j [H(t-t_{j-1}) - H(t-t_j)]$$

Heaviside function

- θ_2 : vector of m sampled values

$$z_i = \frac{1}{\Delta t} \int_{t_{i-1}}^{t_i} z(t) dt \approx \frac{1}{2} (z(t_{i-1}) + z(t_i)) \Rightarrow \text{cond}(\mathbf{M}(\theta_1^{\text{stairs}})) = 3.64 \cdot 10^8$$

for $z = \theta_1$ or W

Other choice (sampled values)

$$z_i = z(t_i) \text{ for } z = \theta_1 \text{ or } W \Rightarrow \text{cond}(\mathbf{M}(\theta_1^{\text{sampled}})) = 2.05 \cdot 10^5$$